

Training Course
Carbon dioxide Refrigeration
R744/CO₂

Overview: Modules and Sections

History and importance of carbon dioxide

Carbon dioxide (CO₂) refrigerant issues

European standards and regulations

CO₂ application potential

Design, Construction, PPM

Safety issues

First Aid

Review of tools and equipment

Commissioning and maintenance

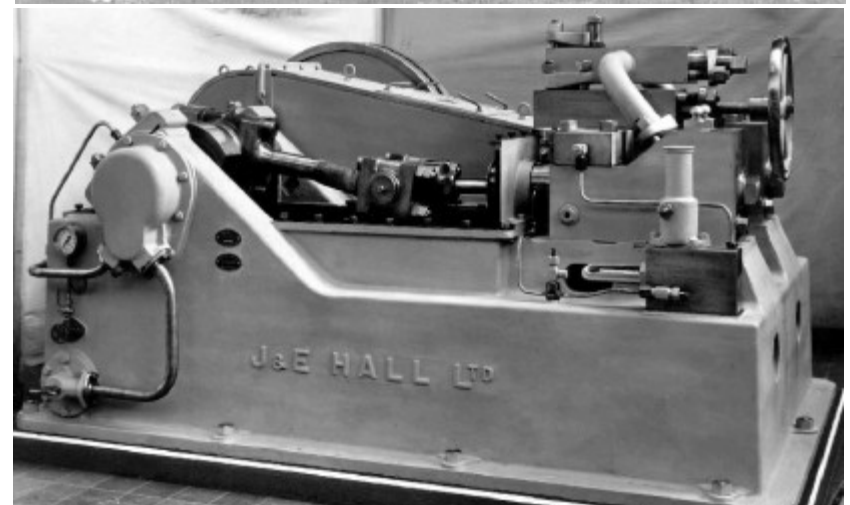
Mechanical integrity

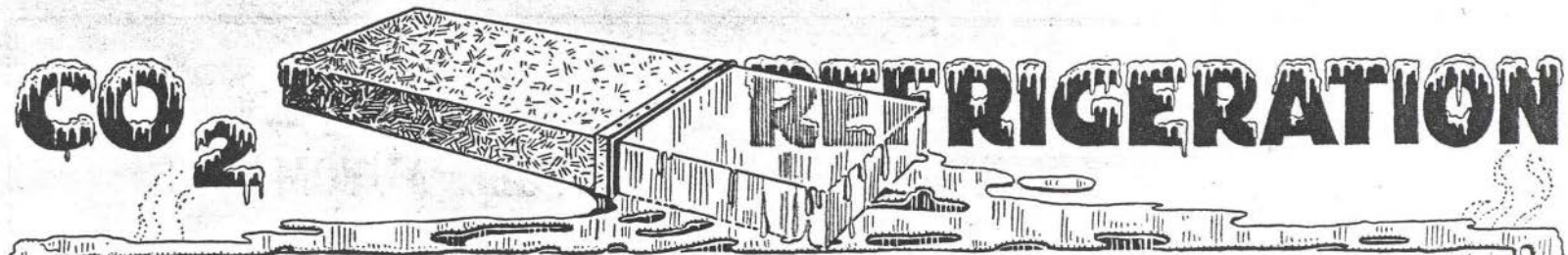
Ice and Refrigeration, 1886



It was reported in The Times in 1886 that J & E Hall's first cold air machines froze 30,000 mutton carcasses being shipped from The Falkland Islands to the UK

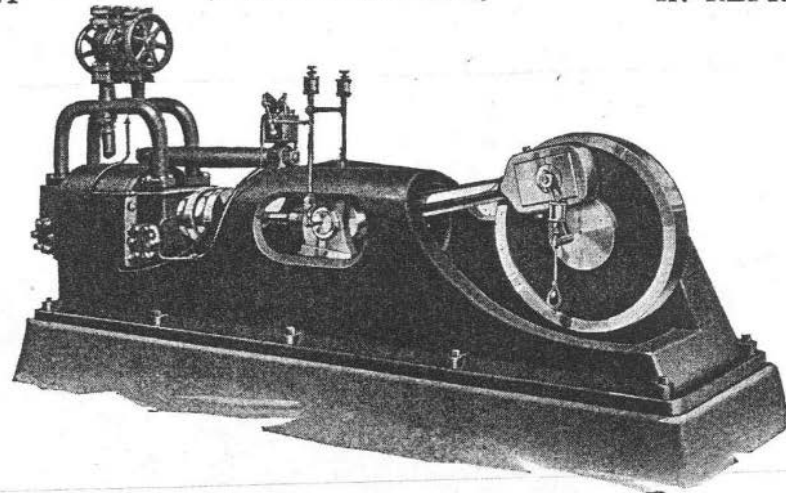
J & E Hall designed the first two stage carbon dioxide compressor in 1886 and, as early as 1910 supplied refrigeration equipment to the National Skating Palace.





CARBONIC SAFETY SYSTEM

MEANS BEST (REG. U. S. PAT. OFF.) IN REFRIGERATION



American Carbonic Machinery Co.

NEW YORK
30 CHURCH STREET

WISCONSIN RAPIDS, WISCONSIN
CLEVELAND
65TH AND EUCLID AVENUE

CHICAGO
1631 MONADNOCK BLDG.

ST. PAUL
43 W. 4th STREET

CARBONIC SAFETY SYSTEM



Beside refrigeration, CO₂ is widely used:

Inert gas: Carbon dioxide is used as an atmosphere for welding

Fire extinguisher: to extinguish flames by flooding the environment around the flame with the gas

As solvent: Liquid carbon dioxide is a good solvent , and is used to remove caffeine from coffee

Beverage industry: transferred into refreshing drinks for sparkling taste

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CO₂ as refrigerant. Advantages and performance. Brief introduction.

- CO₂ is a natural refrigerant with very low global warming potential
- ODP = 0, GWP = 1
- **Non-Toxic, Non-Flammable**
- CO₂ is an inexpensive refrigerant compared with HCFCs and HFCs
- CO₂ has better heat transfer properties compared to conventional HCFCs and HFCs
- More than 50% reduction in HFC refrigerant charge possible (high volumetric cooling capacity)
- CO₂ lines are typically one to two sizes smaller than traditional DX piping systems
- Excellent material compatibility
- System energy performance equivalent or better than traditional HFC systems in cool climates

CO₂ refrigerant issues



CO₂ as refrigerant:

CO₂ is available in different degrees of purity.

It is recommended only to use refrigerant degree of 4.5 or higher.

Classification	CO ₂ "KK"	CO ₂ 3.0	CO ₂ 4.5	R744 (CO ₂ 4.5)
	915	918	925	690
Purity	≥ 99,5	≥ 99,9	≥ 99,995	≥ 99,995
	o. A.	O ₂ + N ₂ ≤ 500	O ₂ ≤ 15	O ₂ ≤ 15
		C _n H _m ≤ 50	N ₂ ≤ 30	N ₂ ≤ 30
		H ₂ O ≤ 250	C _n H _m ≤ 2	C _n H _m ≤ 2
			CO ≤ 1	CO ≤ 1
			H ₂ O ≤ 5	H ₂ O ≤ 5

General refrigerant classification



	Lower (chronic) toxicity		Higher (chronic) toxicity	
No flame propagation	A1	R22 R744 R134a R4010A R404	B1	R123 R245fa
Lower flammability	A2L	R32 R143a R1234yf R1234ze R444A/B	B2L	R717
Flammable	A2	R152a R142b R405A R411A R439A	B2	R30
Higher flammability	A3	R290 R600a R1270 R443A E170	B3	R1140

More onerous requirements

More onerous requirements

EN378-1 Annex E
AHRI 700
ISO817

Flammability and toxicity classification



Safety classification	Lower flammability level, % in air by volume	Heat of combustion J/kg	Flame propagation
A1	No flame propagation when tested at 60°C and 1013 mbar (101.3 kPa)		
A2 lower flammability	> 3.5	< 19,000	Exhibit flame propagation when tested at 60°C and 1013 mbar (101.3 kPa)
A2L lower flammability	> 3.5	< 19,000	Exhibit flame propagation when tested at 60°C and 1013 mbar (101.3 kPa) and have a maximum burning velocity of ≤ 10 cm/s when tested at 23°C and 1013 mbar
A3 higher flammability	≤ 3.5	≥ 19,000	Exhibit flame propagation when tested at 60°C and 1013 mbar

CO₂ refrigerant issues



REFRIGERANT	REFRIGERANT NUMBER	CHEMICAL FORMULA	GWP (100 YEARS)	ODP	NORMAL BOILING POINT (°C)	CRITICAL TEMPERATURE (°C)	CRITICAL PRESSURE (BAR)	SAFETY GROUP	MOLECULAR WEIGHT (G/MOL)
Ammonia	R717	NH ₃	0	0	-33.3	132.4	114.2	B2L	17.03
Carbon dioxide	R744	CO ₂	1	0	-78	31.4	73.8	A1	44.0
Propane	R290	C ₃ H ₈	3.3	0	-42.1	96.7	42.5	A3	44.10
Isobutane	R600a	C ₄ H ₁₀	4	0	-11.8	134.7	36.48	A3	58.12
Propylene	R1270	C ₃ H ₆	1.8	0	-48	91	46.1	A3	42.08
Water	R718	H ₂ O	0	0	100	373.9	217.7	A1	18.0
Air	R729	-	0	0	-192.97	-	-	-	28.97

Comparison of properties



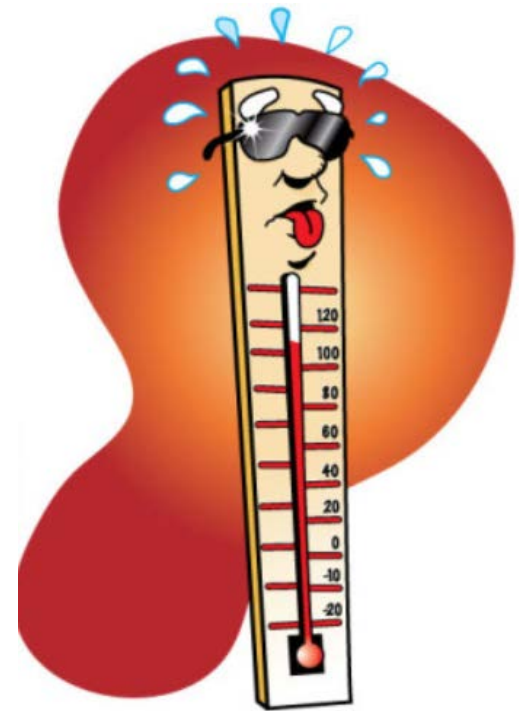
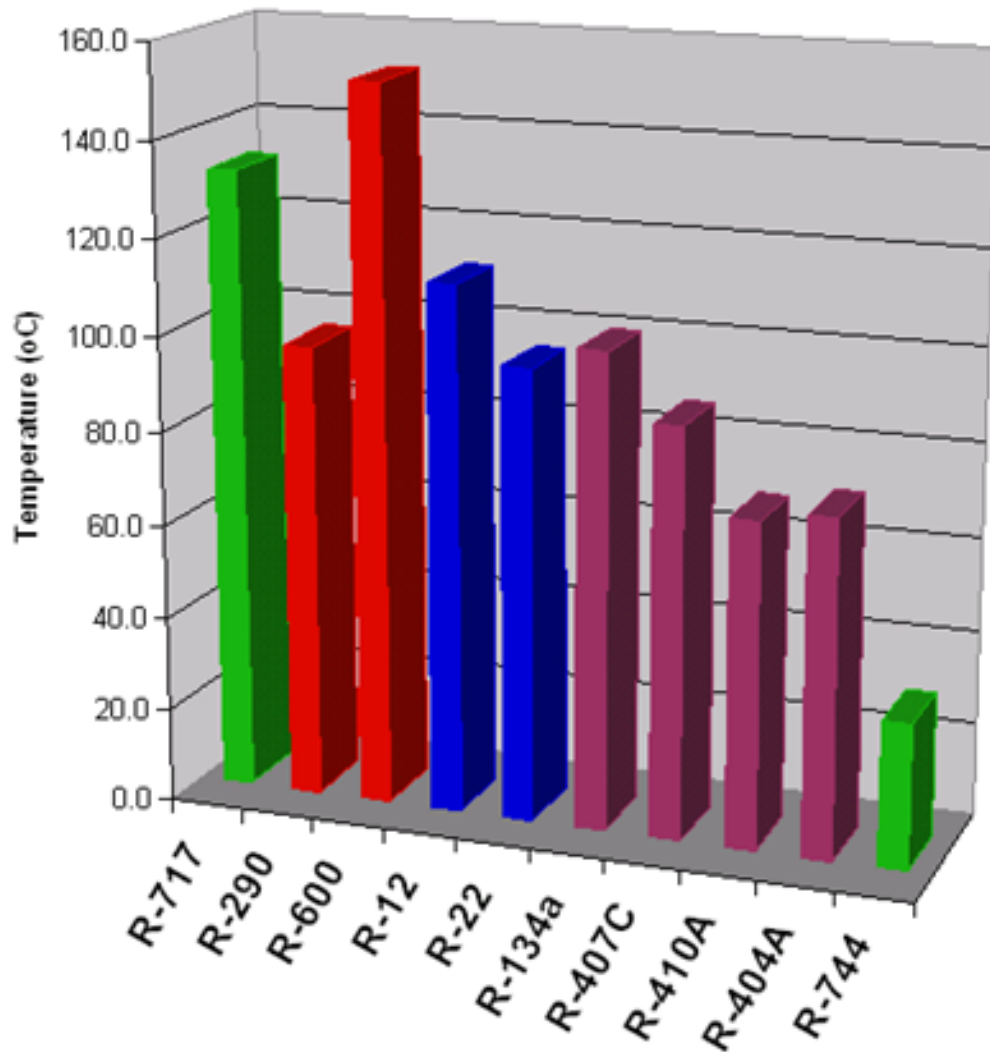
Refrigerant	R744	R404A	R134a	R407A	R407F
Temperature at atmospheric pressure	-109.3 °F (-78.5 °C) Temp. of dry ice	-50.8 °F (-46 °C) (Saturation temp.)	-14.8 °F (-26 °C) (Saturation temp.)	-41.8 °F (-41 °C) (Mid-point saturation temp.)	-45.5 °F (-43 °C) (Mid-point saturation temp.)
Critical temperature	87.8 °F (31 °C)	161.6 °F (72 °C)	213.8 °F (101 °C)	179.6 °F (82 °C)	181.4 °F (83 °C)
Critical pressure	1,056 psig (72.8 barg)	503 psig (34.7 barg)	590 psig (40.7 barg)	641 psig (44.2 barg)	674 psig (46.5 barg)
Triple point pressure	60.6 psig (4.2 bar abs)	0.44 psig (0.03 bar abs)	0.0734 psig (0.005 bar abs)	0.19 psig (0.013 bar abs)	---
Pressure at a saturated temperature of 20 °C (68 °F)	815 psig (56.2 barg)	144 psig (9.9 barg)	68 psig (4.7 barg)	133 psig (9.2 barg)	139 psig (9.6 barg)
Global warming potential	1	3922	1430	1990	1824



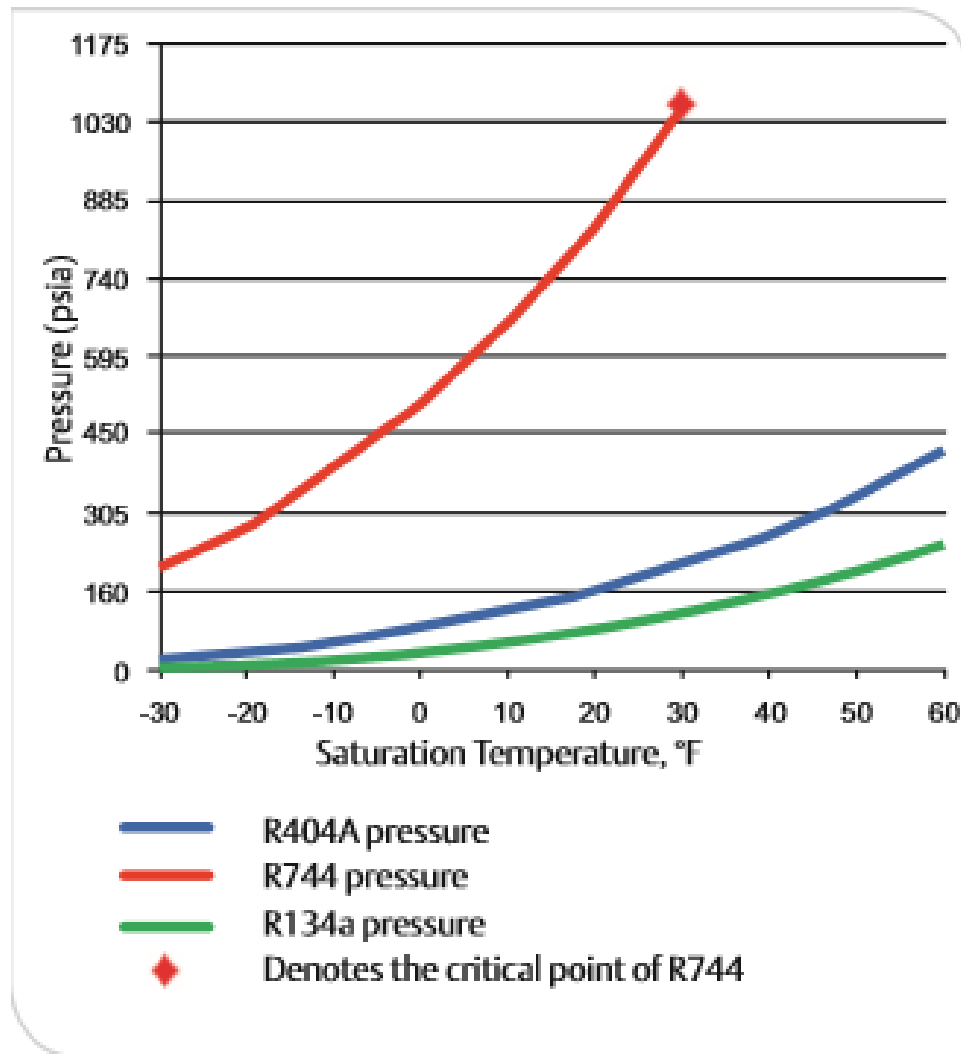
Comparison of properties

Refrigerant	Critical temp. (°C)	Liquid viscosity (Pa s ×10 ⁶)	Vapour viscosity (Pa s ×10 ⁶)	Liquid sp. heat (kJ/kg K)	Liq thermal cond (W/m K)	Latent heat (kJ/kg)
R-22	96.1	216	11.4	1.17	0.095	205
R-134a	101.1	267	10.7	1.34	0.092	199
R-404A	72.0	179	11.0	1.39	0.073	166
R-407C	86.0	211	11.3	1.42	0.096	210
R-410A	71.4	161	12.2	1.52	0.103	221
R-717	132.3	170	9.1	4.62	0.559	1262
R-290	96.7	126	7.4	2.49	0.106	375
R-1270	92.4	121	7.8	2.44	0.126	378
R-744	31.0	99	14.8	2.54	0.110	231

Critical Temperatures



Critical Pressure

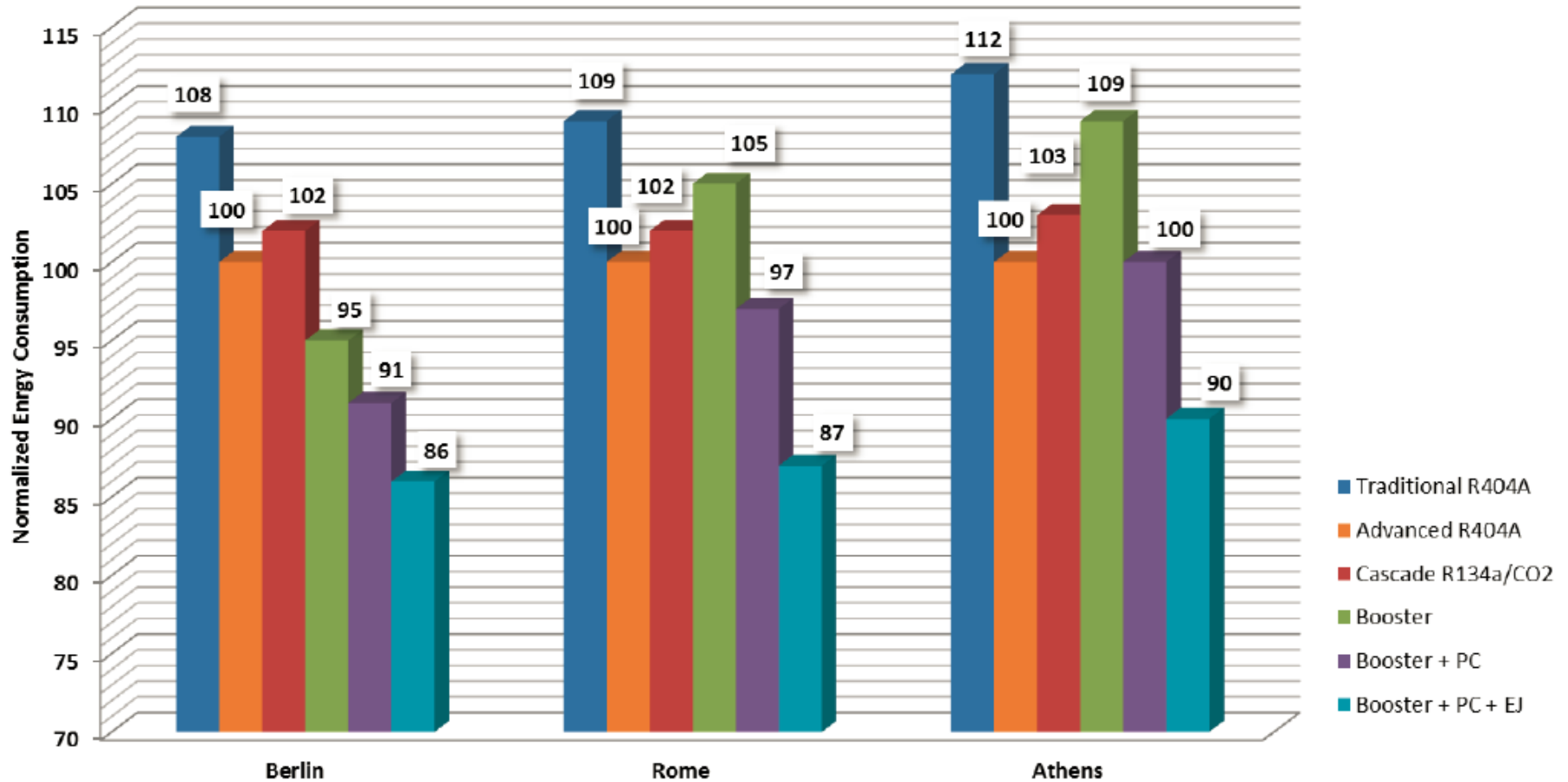


$$t_{\text{crit}} = 31^{\circ}\text{C}$$

$$p_{\text{crit}} = 73,77 \text{ bar}$$

Figure 5. Pressure-temperature relationship comparison

Performance analysis



General Servicing Issues	NH3 R717	CO ₂ R744	HCs R290, R600a ...
Weight in relation to air	Lighter	Heavier	Heavier
Refrigerant Purity	99.98 % min Moisture < 200 ppm	99,99 % Moisture < 10 ppm	99.5 % min Moisture < 10 ppm
Gauges & Circuit Equipment	Stainless Steel R717 indication	High Pressure R744 indication	As for HCFC / HFC HC indication
Vacuum Pump	Stainless Steel ATEX, Vent Line	Regular Vent line	Regular ATEX, Vent line
Charging	Scale	Scale Pressure	Sensitive Scale
Tubing	Carbon steel, stainless steel	Copper HP Stainless steel	Copper
Leak Finding	Nose, Gas detector, Litmus paper, Sulfur stick, Bubble test, PPE	Gas detector, Bubble test, PPE	Gas detector Bubble test, PPE
Pressure test Leak Test	Nitrogen 4.0	Nitrogen 4.0 Trace Gas (N ₂ /H ₂)	Nitrogen 4.0 Trace Gas (N ₂ /H ₂)
Strength test PS x 1.1	Nitrogen 4.0	Nitrogen 4.0	Nitrogen 4.0

HP copper examples:

Heat treatable copper-iron alloys have a higher strength than copper whilst maintaining a reasonably high electrical and thermal conductivity and excellent welding, soldering and brazing properties. **Pressure (PS) = up to 120bar**



CO₂ > Conclusions

CO₂ is safe if it is used correctly

It is easy to use, and reliable in operation

It offers high efficiency

The two key drivers for increased adoption are:

- Safe design of piping and vessels (pressure)
- Enlightened regulation

Incentive will come from:

- Training and familiarity
- Technical Development
- Economies of Scale



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Important Rules, Regulations, Laws

- DGUV rule 100-500 > prevention of accidents
- PED European pressure equipment directive
- Machinery directive 95/16/EC
- CEN EN 378/2016 (part 1-4)
- DIN 2405 Pipelines in refrigerating systems and cooling equipment - Marking EN 378 Part 1 to 4 (new release early 2017)

Design Considerations: Recommendations



Purchaser:

- Specify EN378 as minimum design
- Detectors and alarms
- Automatic plant room ventilation

Designer/Installer:

- Ensure compliance with EN378 parts 2 and 3
- Pipe and components designed for high pressure
- Design for lifetime operation and efficiency

Operator:

- Ensure that the plant is designed for maintenance

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- Several companies producing
 - Air-cooled and water-cooled R744 (CO2) chillers
- Safety aspects
 - Designed to EN 378
- Cost more than HFC/HCFC
- R744 gives higher efficiency than HFC/HCFC in cooler climates



Typical transcritical booster systems



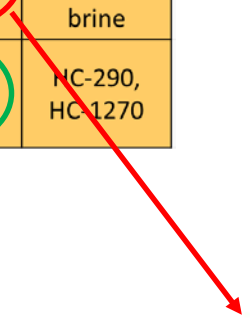
R744 Cascade, Booster systems



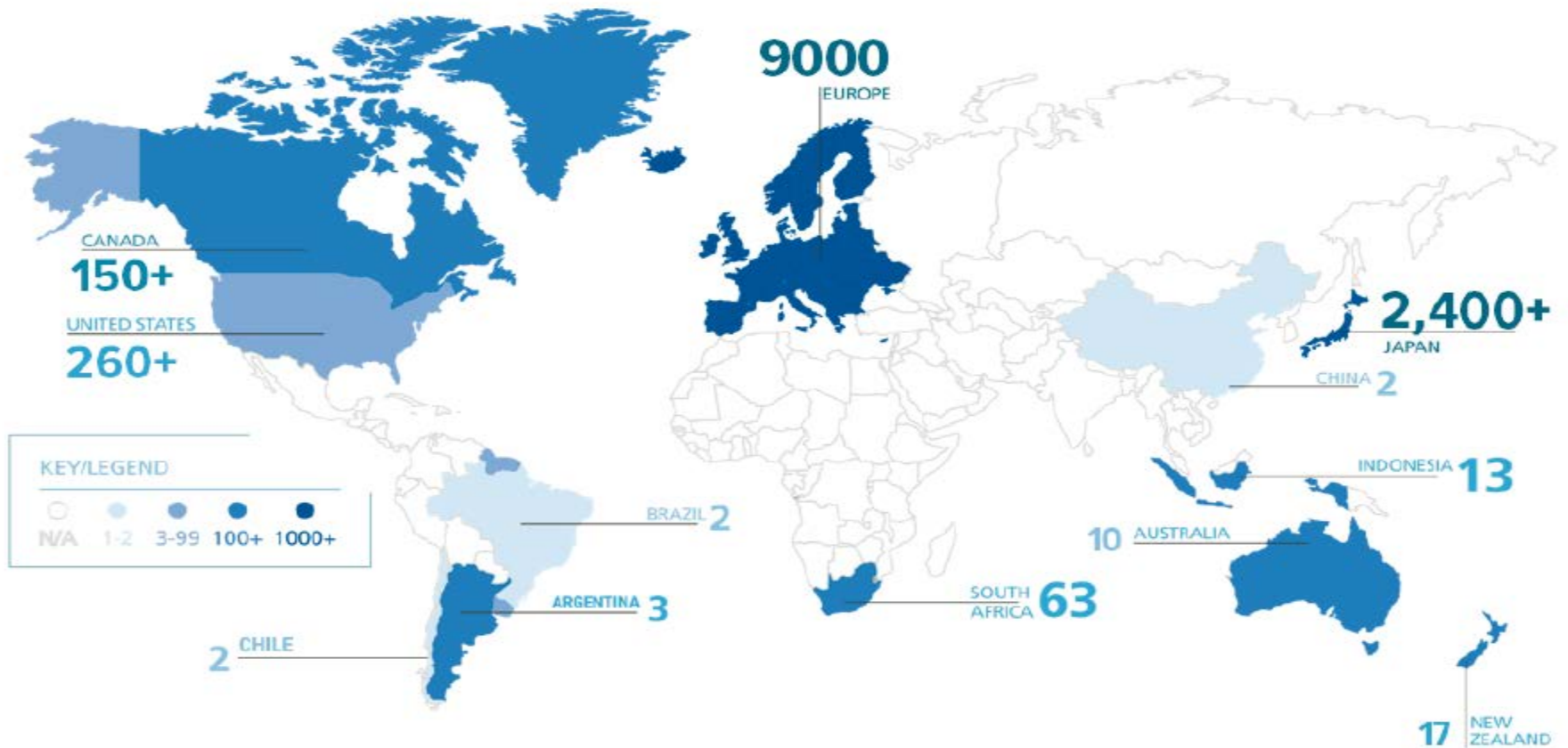
Example of products in the commercial sector



	Conventional	Alternative				
Type	Direct expansion, multi-compressor pack	Cascade	Indirect (liquid sec)/ cascade	Indirect (phase-change sec)/ cascade	Trans-critical booster	Distributed water cooled
Medium temp	R404A, R507A, R407F	Lower GWP HFC, (HC-290, HC-1270)	HC-290, HC-1270, R-717, brine	HC-290, HC-1270, R-717, CO2	R744	HC-290, HC-1270, R-717, brine
Low temp	R404A, R507A, R407F	R744	CO2	CO2	R744	HC-290, HC-1270



CO2 TC stores growing globally

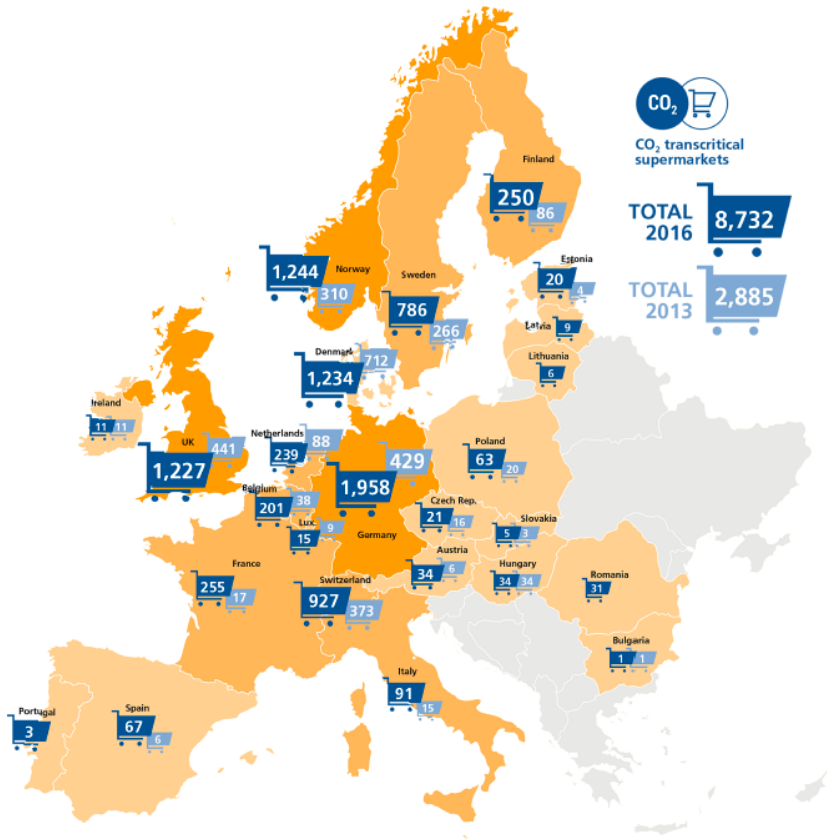


Source: Shecco, 2017

CO2 TC stores growing globally



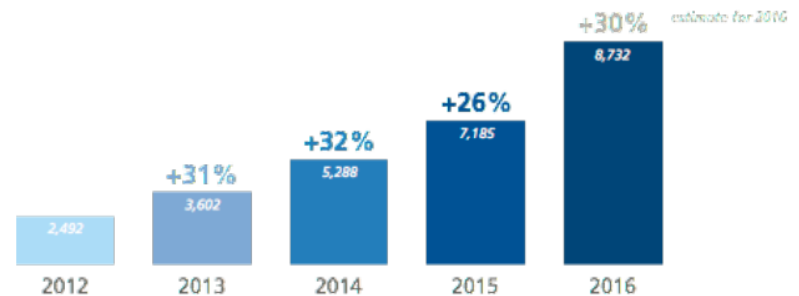
EU Stores using cutting-edge HFCfree technology



Amount of CO₂ stores in the EU, Norway, Switzerland has tripled in the last 3 years = 8% of the overall market share in the food retail market

New installations is growing steeply in Southern Europe

Growth of CO₂-based stores



Source: Shecco, 2017

Example of R744 condensing units



Suited to small stores: compact, CO2 TC units, with high efficiency

- E.g., Advansor
 - Range of R744 (CO2) condensing units
- Safety aspects
 - Designed to EN 378
- Cost more than HFC
- R744 gives higher efficiency than HFC
 - Less likely in hotter climates



Model: compSUPER	XXS 1x0MT4	XXS 1x0MT5	XXS 1x0MT7	XXS 1x0MT9	XXS 0x1 LT4	XXS 0x1 LT16
MT/LT compressors	1/0	1/0	1/0	1/0	0/1	0/1
Frequency range	30/87Hz	30/87Hz	30/87Hz	30/87Hz	35/70 Hz	50/70 Hz
Cooling capacity [kW]*	1,6/4,3	2,1/5,6	2,7/7,2	3,4/9,0	2,2/4,2	12,3/16,1

Application of R744 plug-ins

- Coca Cola adopted “cassette” concept for bottle coolers and vending machines
 - The components of a R744 refrigeration system are similar to those used in an R134a system

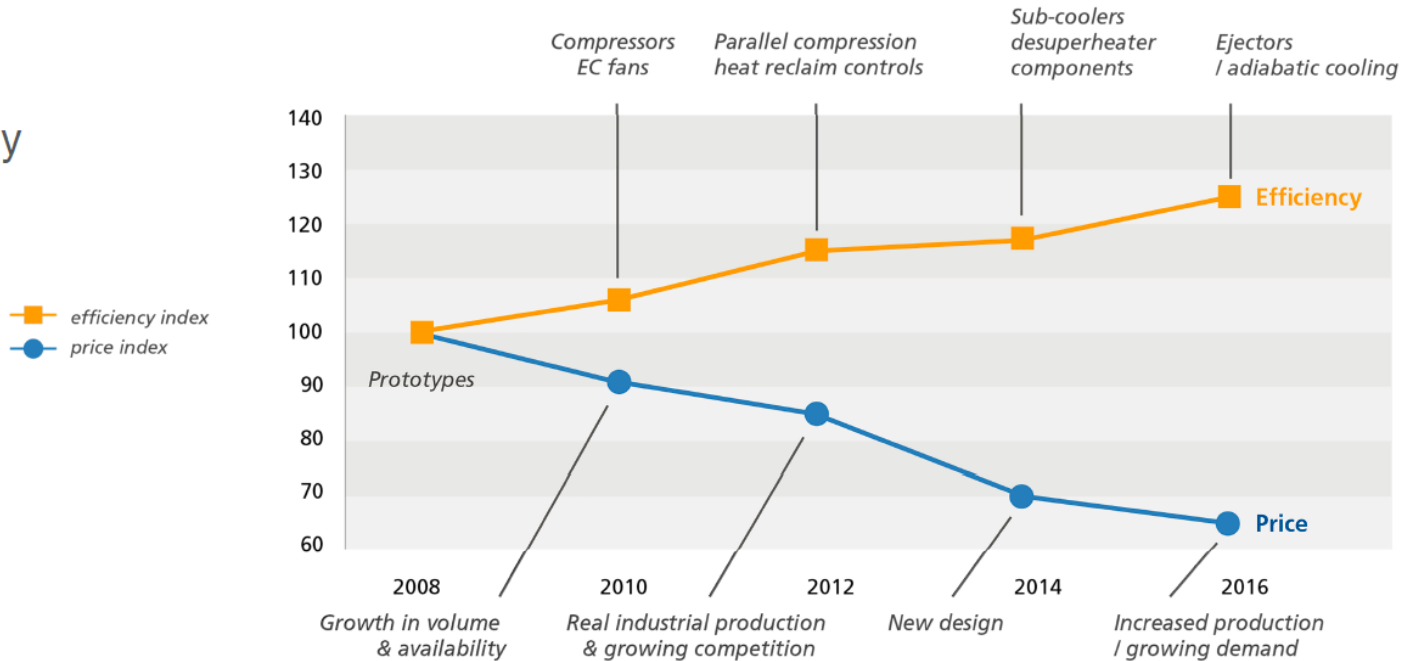


CO2 cost of equipment decreasing



Higher energy efficiency
but lower prices

In Commercial
Refrigeration, price is
the same as HFC
technology or 5-10%
higher



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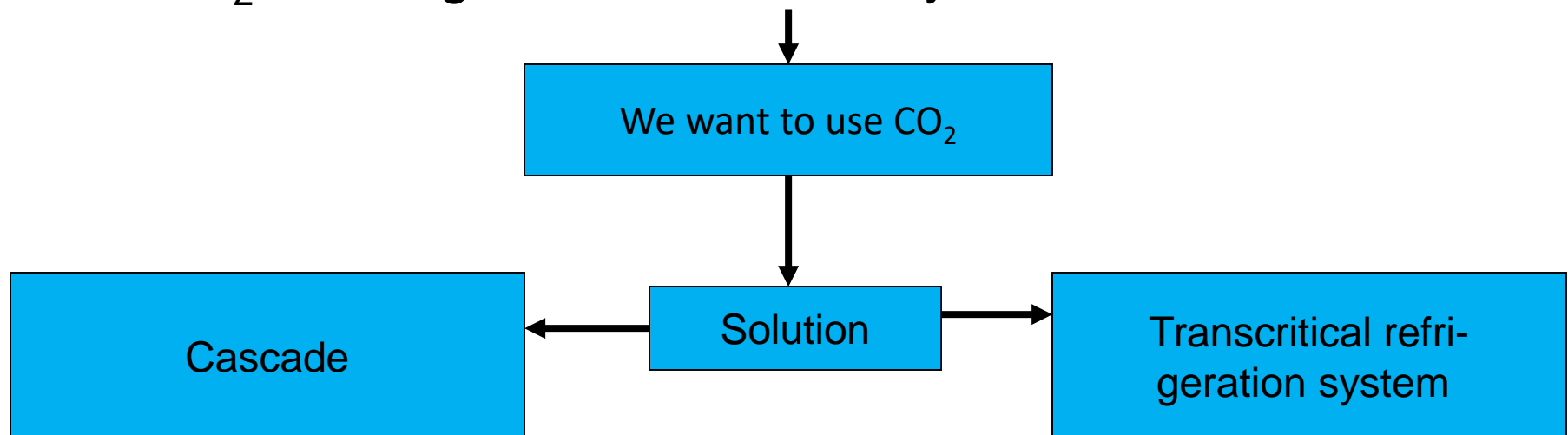
The critical point of CO₂



CO₂ has a critical temperature of $t_{\text{crit}} = 31^\circ \text{C}$ and a critical pressure of $p_{\text{crit}} = 73,77 \text{ bar}$.

With an ambient temperature in summer of $t_{\text{ambient}} = 35^\circ \text{C}$ and as a result from $\Delta T = 10 \text{ K}$ the „condensing“ temperature is $t_c = 45^\circ \text{C}$.

With CO₂ as refrigerant this is already transcritical.



High Pressure



Comparison of typical pressures in CO₂ systems to those in R404A systems.

	CO₂	R404A
Evaporating pressure	10 to 30 bar g	0.5 to 4 bar g
Condensing (gas cooler) pressure	35 bar g cascade Up to 120 bar g trans critical	Up to 25 bar g
Standstill pressure	Up to 75 bar g	Up to 14 bar g

R744 system sizing



- Higher gas density of R744 results in high volumetric refrigeration effect compared to all other refrigerants. This has an effect on compressor displacement and pipe sizing, evaporators and condensers.

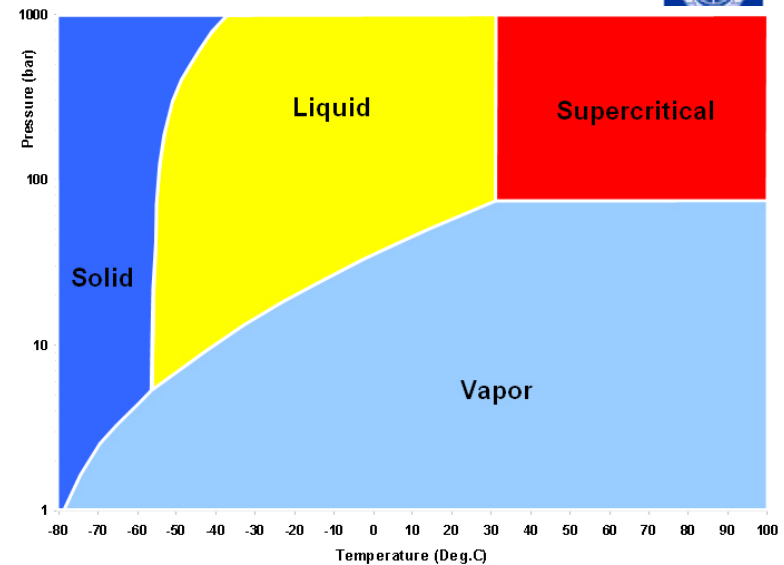
	R404A	R744
Suction line diameter	28 mm	12 mm
Liquid line diameter	8 mm	6 mm

- Power consumption for a given capacity is similar to that of HCFC (HFCs).

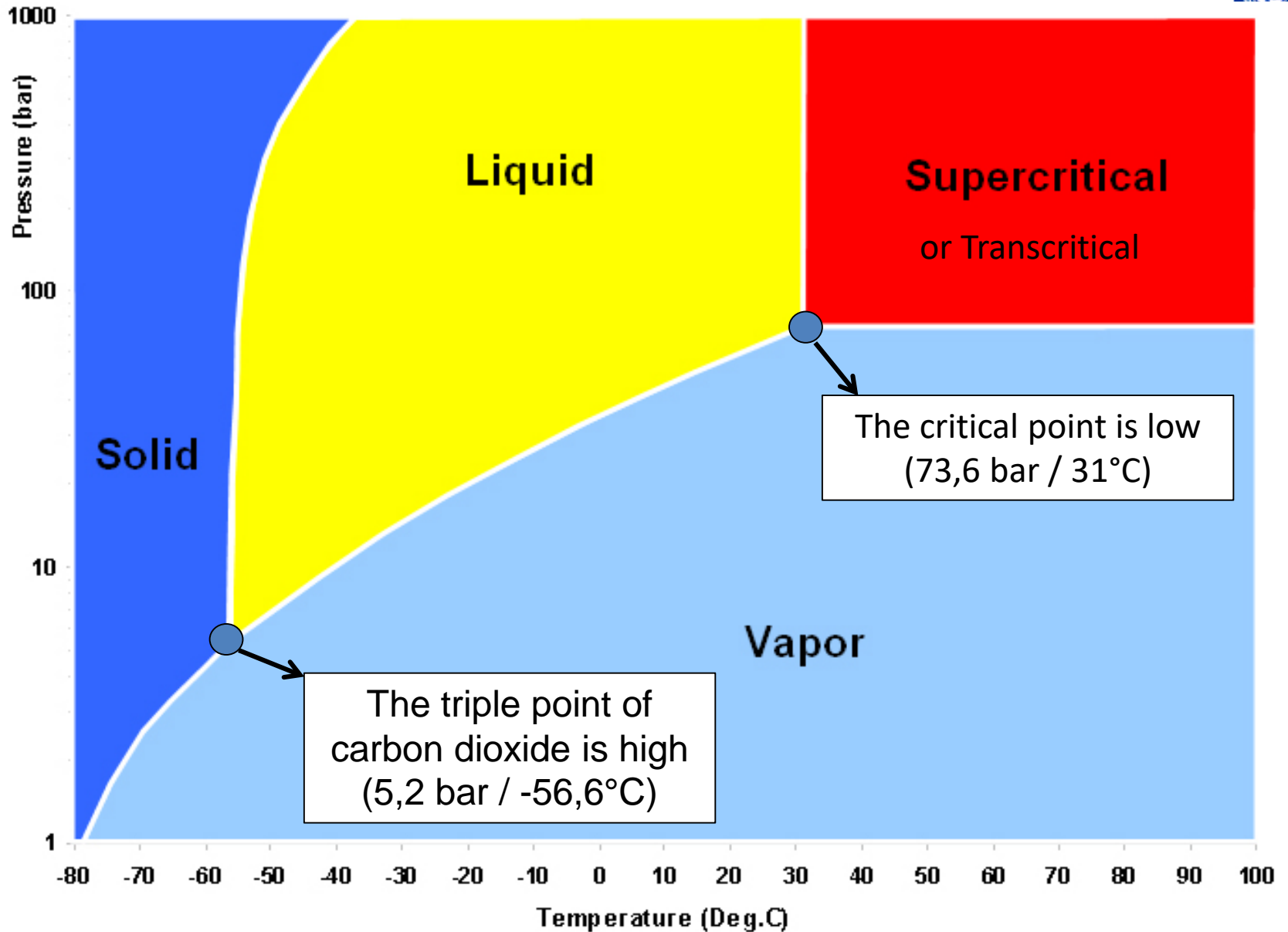
Basic Consideration for Service and Design



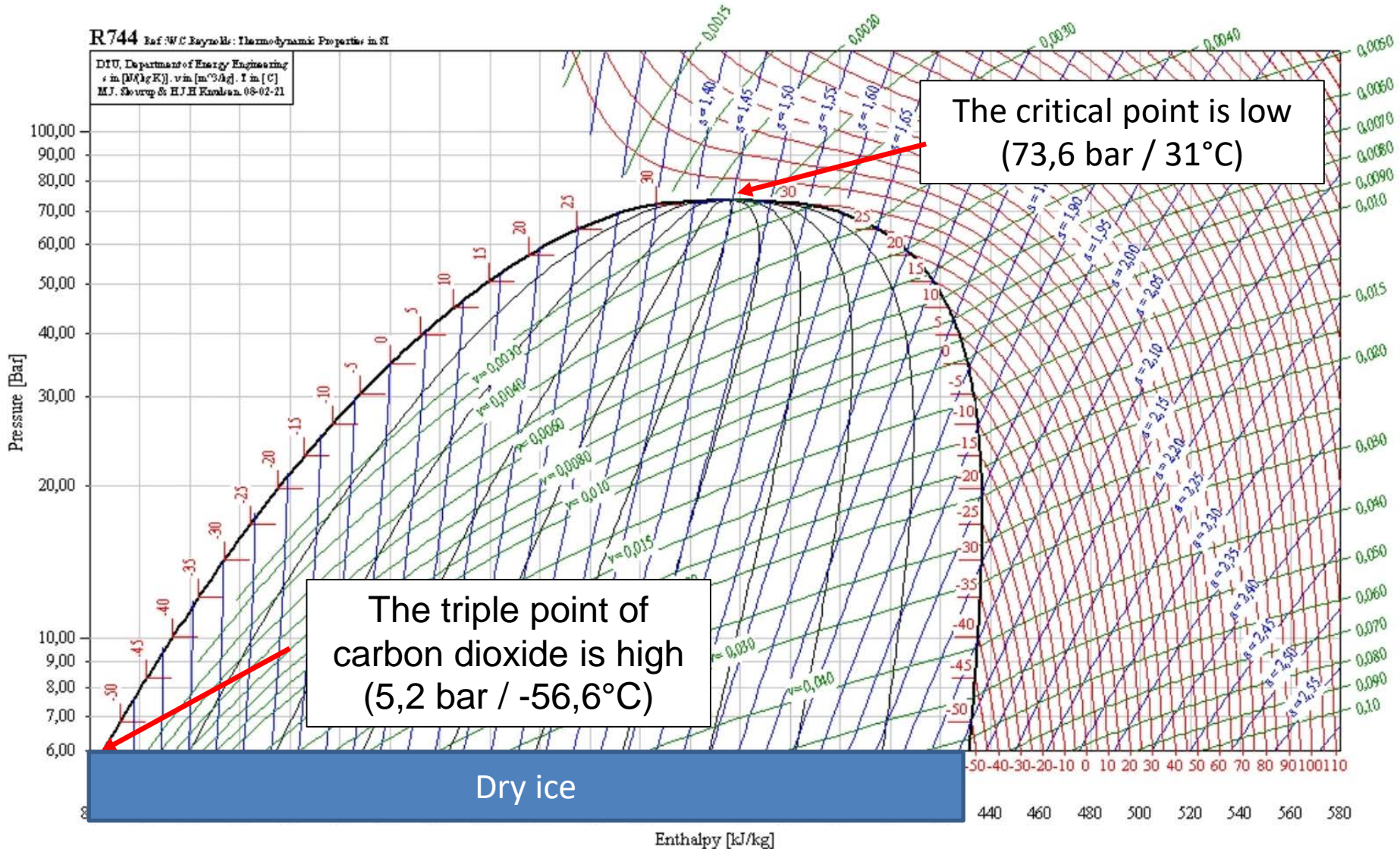
- The critical point is the condition at which the liquid and gas densities are the same. Above this point distinct liquid and gas phases do not exist.
- The triple point is the condition at which solid, liquid and gas coexist.
- The triple point of carbon dioxide is high (5,2 bar / -56,6°C) and the critical point is low (73,6 bar / 31°C) compared to other refrigerants.



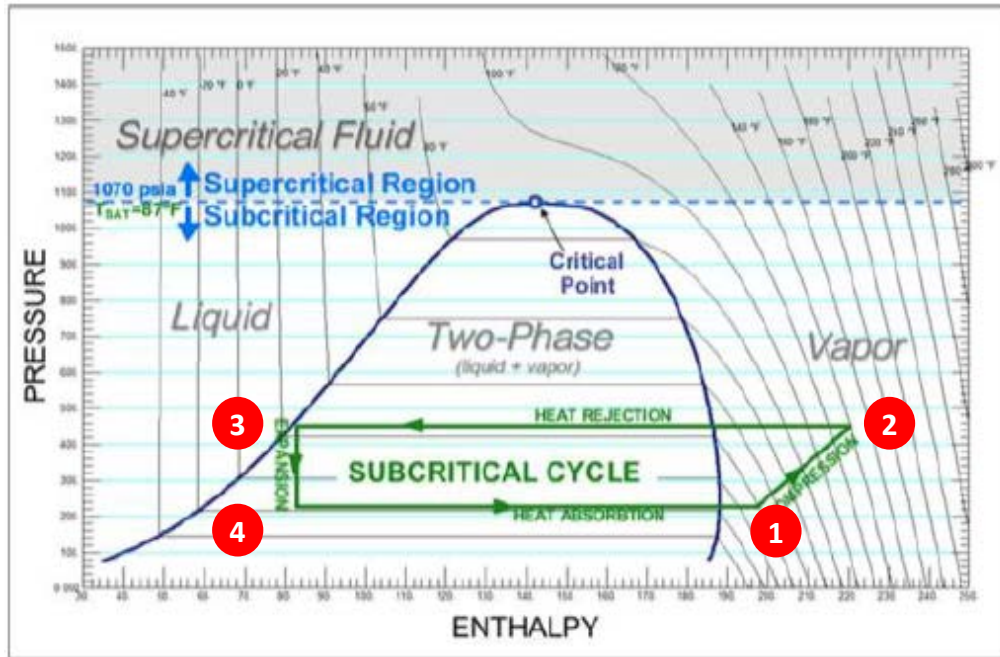
CO2 Phase Diagram



CO2 Phase Diagram

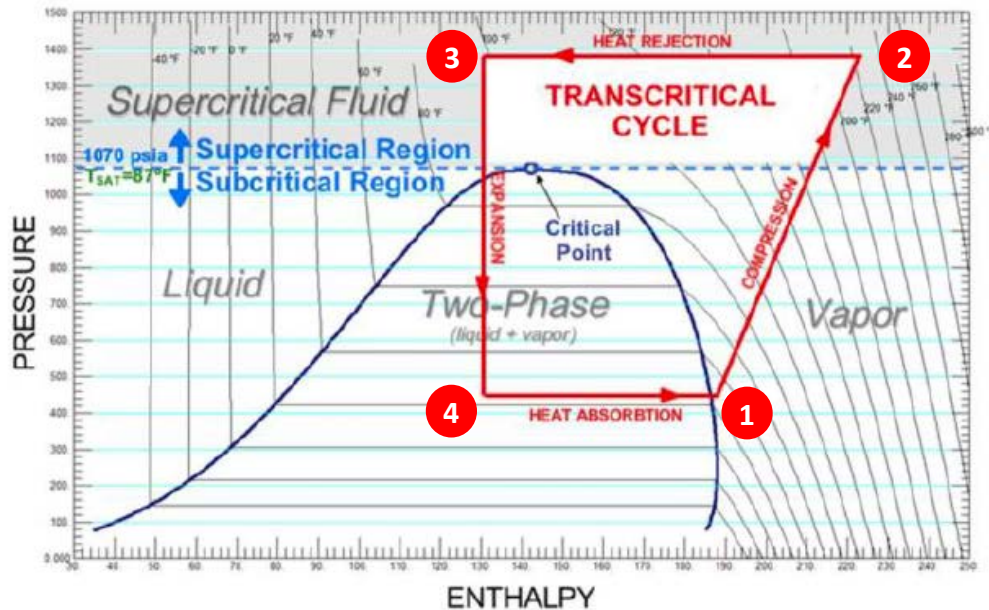


Single-stage subcritical process



- 1-> 2 Compression
- 2-> 3 Condensation
- 3-> 4 Expansion
- 4-> 1 Evaporation

Single-stage transcritical process



- 1-> 2 Compression
- 2-> 3 Heat rejection
- 3-> 4 Expansion
- 4-> 1 Evaporation

Two-stage transcritical process

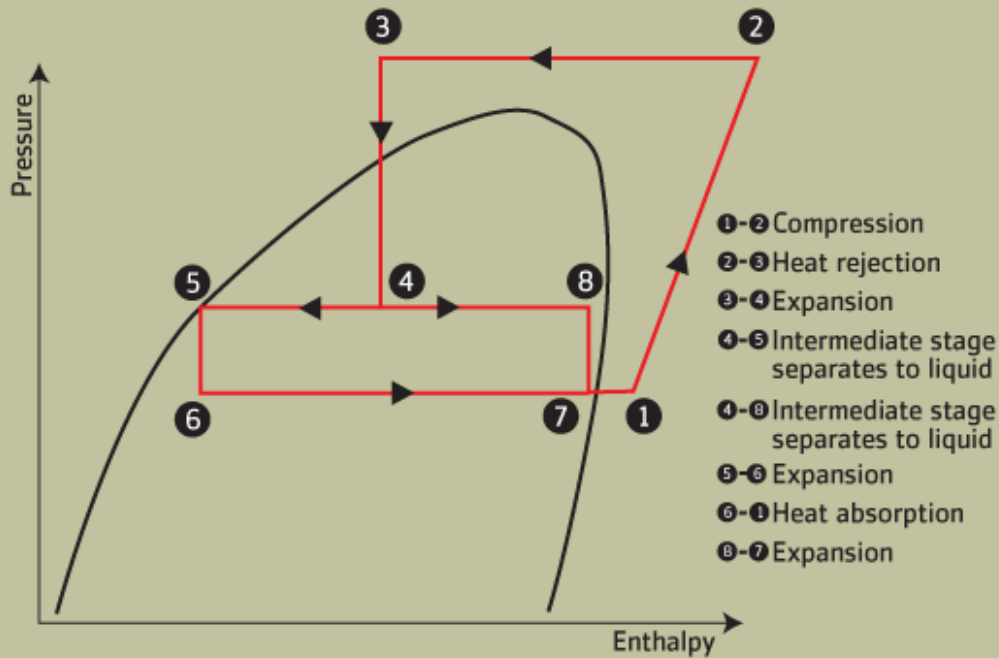
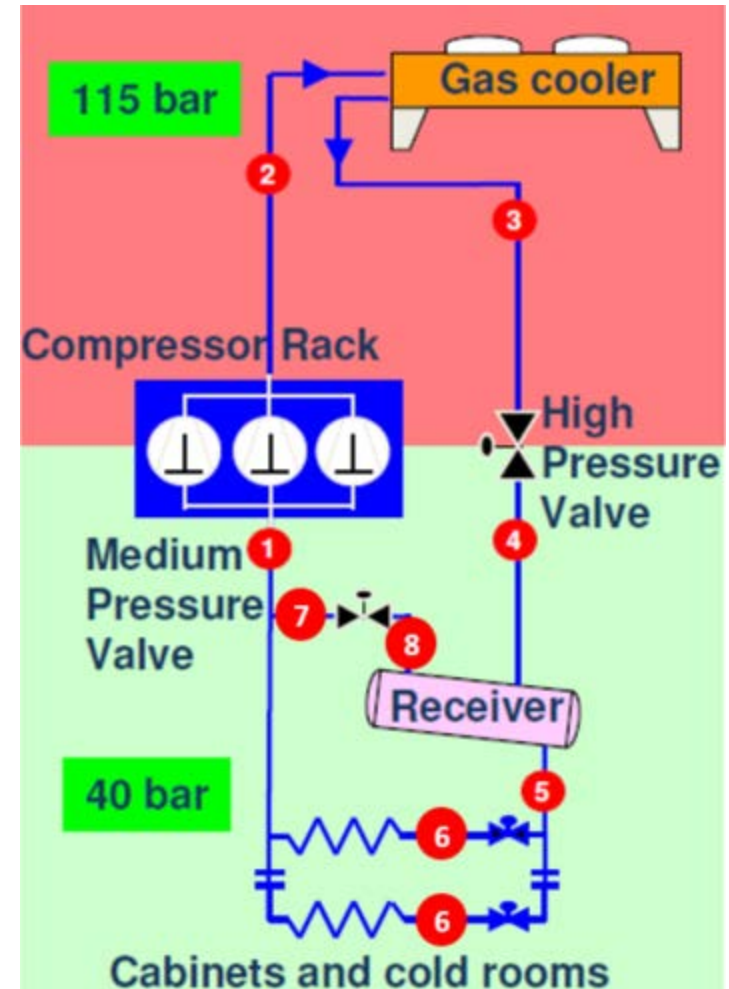


FIGURE 5: Pressure-enthalpy chart of a transcritical CO₂ refrigeration system



Two-stage transcritical process



Compression (1-> 2)

Transcritical approx. 102_bar(g)
Subcritical min.ca. 45bar(g)

Gas cooling (2 -> 3)

Cooling to gas cooler end temperature.
Transcritical approx. 2K above ambient temperature.
(with sprayed water: approx. 3 to 6K below ambient temperature)
Subcritical 3K subcooling

Expansion (3-> 4)

Transition from high to intermediate pressure level.
Intermediate pressure approx. 35 bar(g) (approx. 0° C)

Separation (4-> 5)

Liquid separates from gas within intermediate vessel.

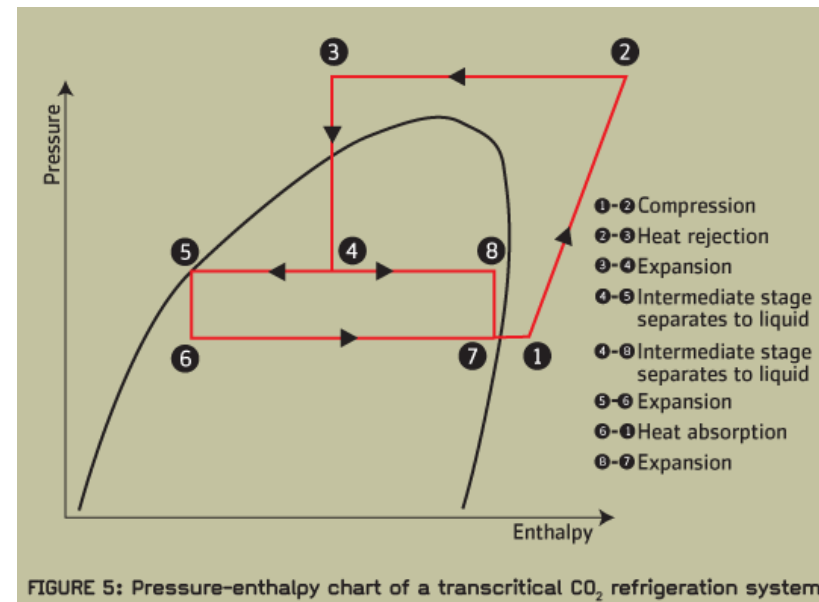
Expansion (5-> 6)

CO₂ expands into evaporator (AKV)

Evaporation and superheating (6-> 7)

Gas throttling (8-> 7)

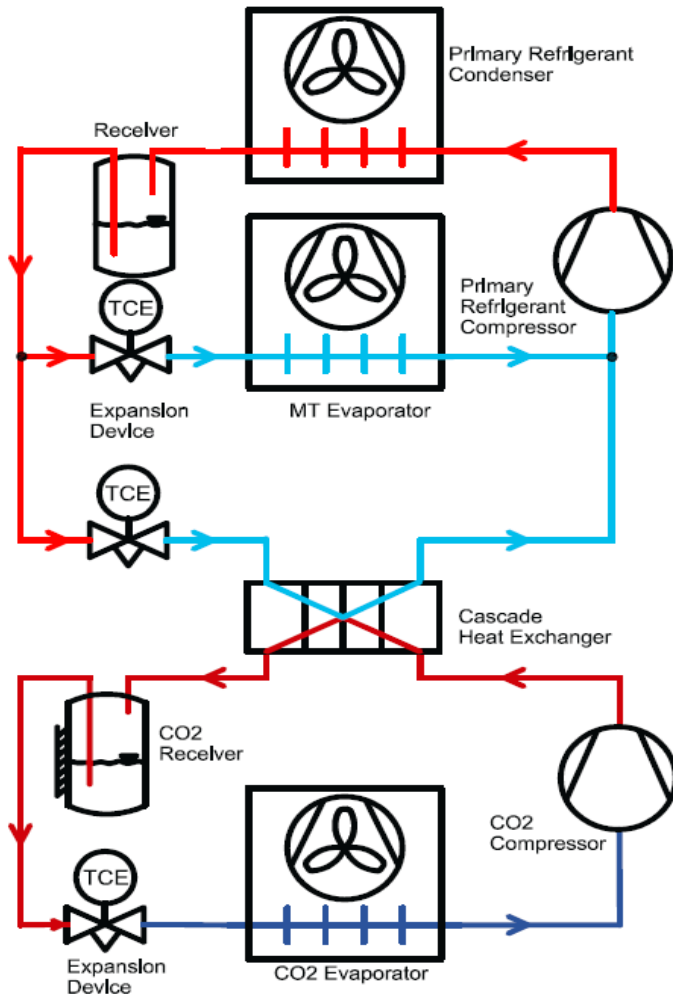
From intermediate pressure to evaporation pressure.
Intermediate pressure regulation (Flash gas)



CO₂ system configurations



Simple cascade system



The cascade system comprises:

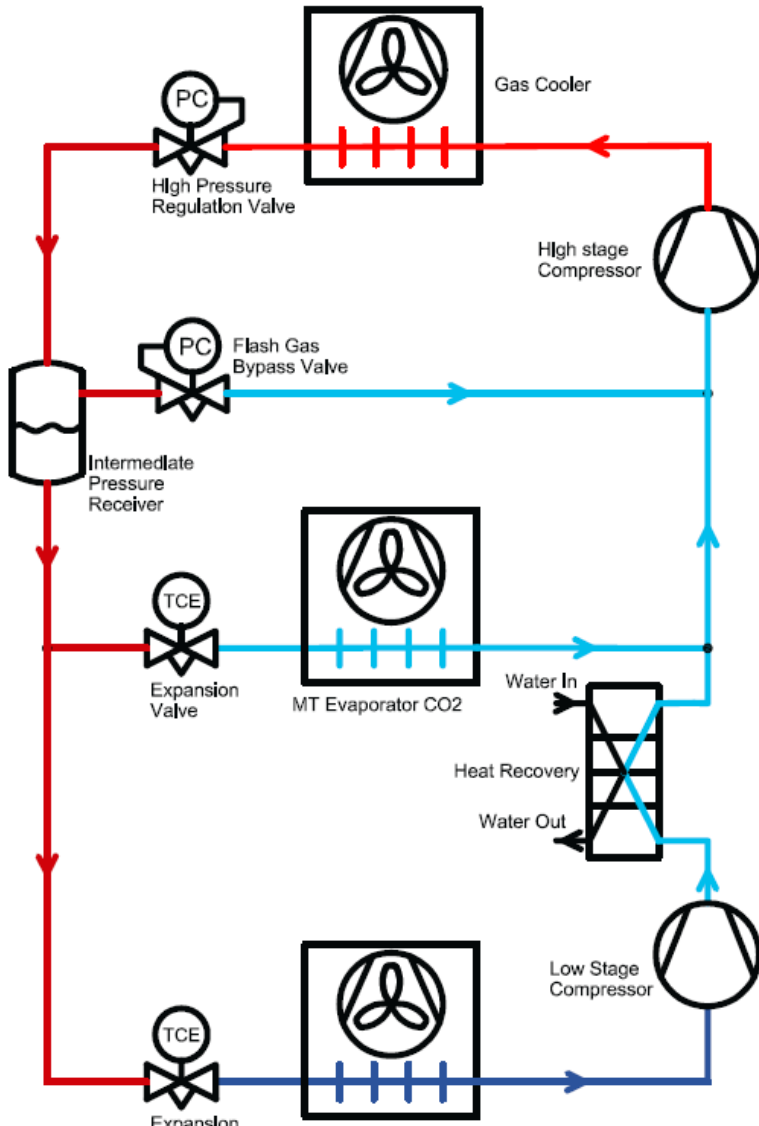
- The low-stage, which provides the cooling for the load. It uses R744 and is always subcritical
- The high-stage, which absorbs the heat from the condensing R744 at the cascade heat exchanger

Within the cascade heat exchanger the evaporating high-stage refrigerant absorbs the heat rejected by the condensing R744.

The R744 condensing temperature is maintained below the critical point.

The high-stage refrigerant is usually an HFC or HC, in which case the cascade is a hybrid system. In some systems R744 is used in the high-stage.

Simple booster system without oil management

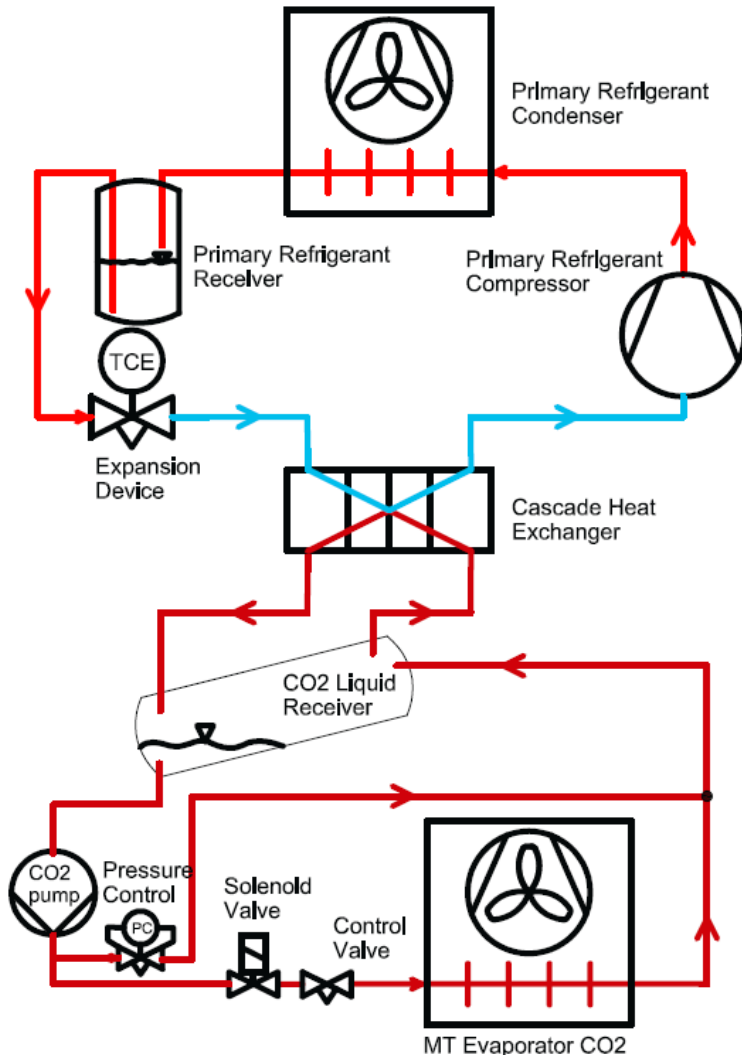


- Two-stage compression is used for transcritical low temperature applications because the discharge temperature of R744 is high and will potentially result in lubricant break down.
- The refrigerant from the low temperature loads is drawn into the low-stage compressors. The discharge from these compressors goes into the suction of the high-stage compressors.
- The refrigerant from the medium temperature (MT) loads is drawn into the suction of the high-stage compressors. The refrigerant from the receiver pressure regulating valve is also drawn into the suction of the high-stage compressors.

CO₂ system configurations



Simple pump system secondary



The high-stage system cools the liquid R744 in the secondary circuit.

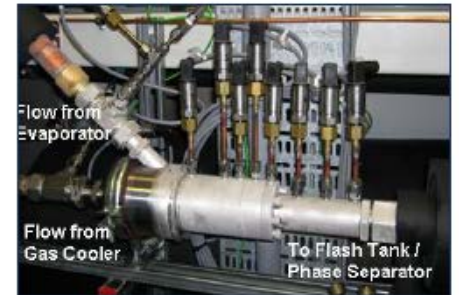
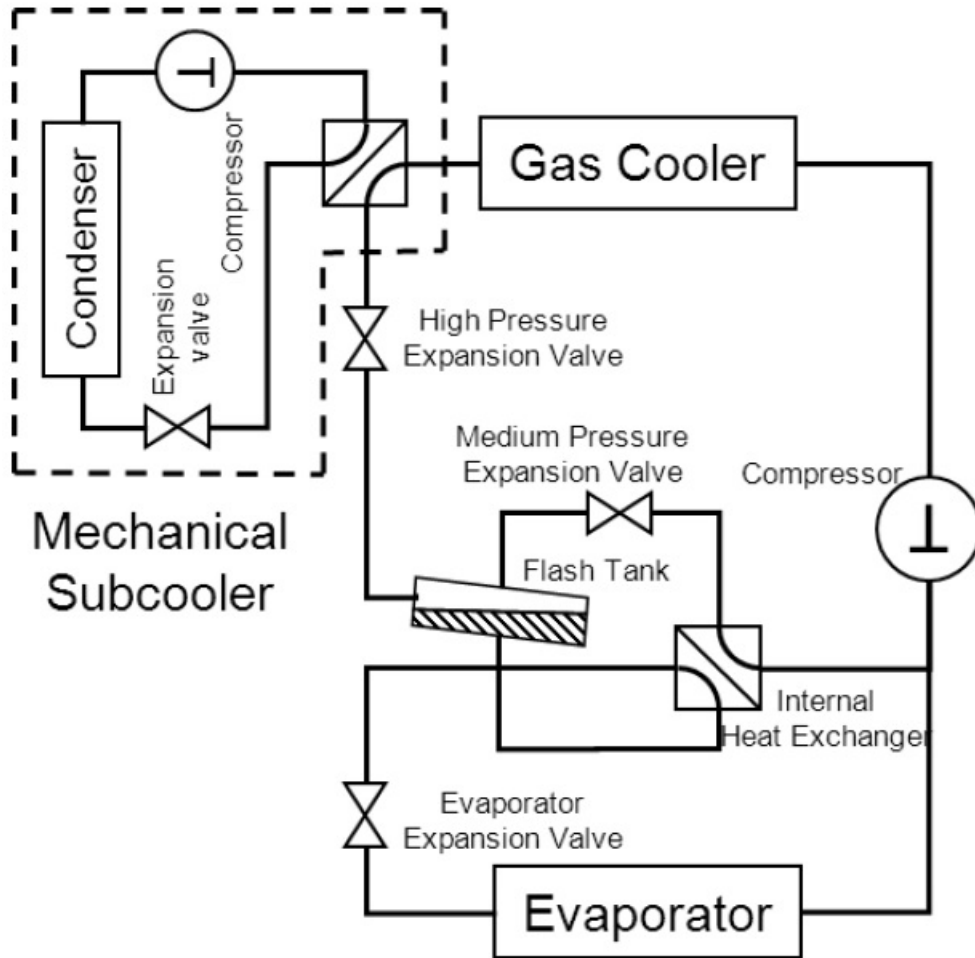
The R744 is pumped around the load. It is volatile, so unlike a conventional secondary fluid such as glycol it does not remain as a liquid, instead it partially evaporates.

It therefore has a significantly greater cooling capacity than other secondary fluids. This reduces the pump power required and the temperature difference needed at the heat exchanger.

The R744 would typically be cooled to -3°C for the MT load, and to -25°C for the LT load.

The high-stage system is a simple chiller type system, typically running on an HFC or HC refrigerant.

For higher ambient temperatures also a mechanical sub-cooler offers good potentials, based on components available today

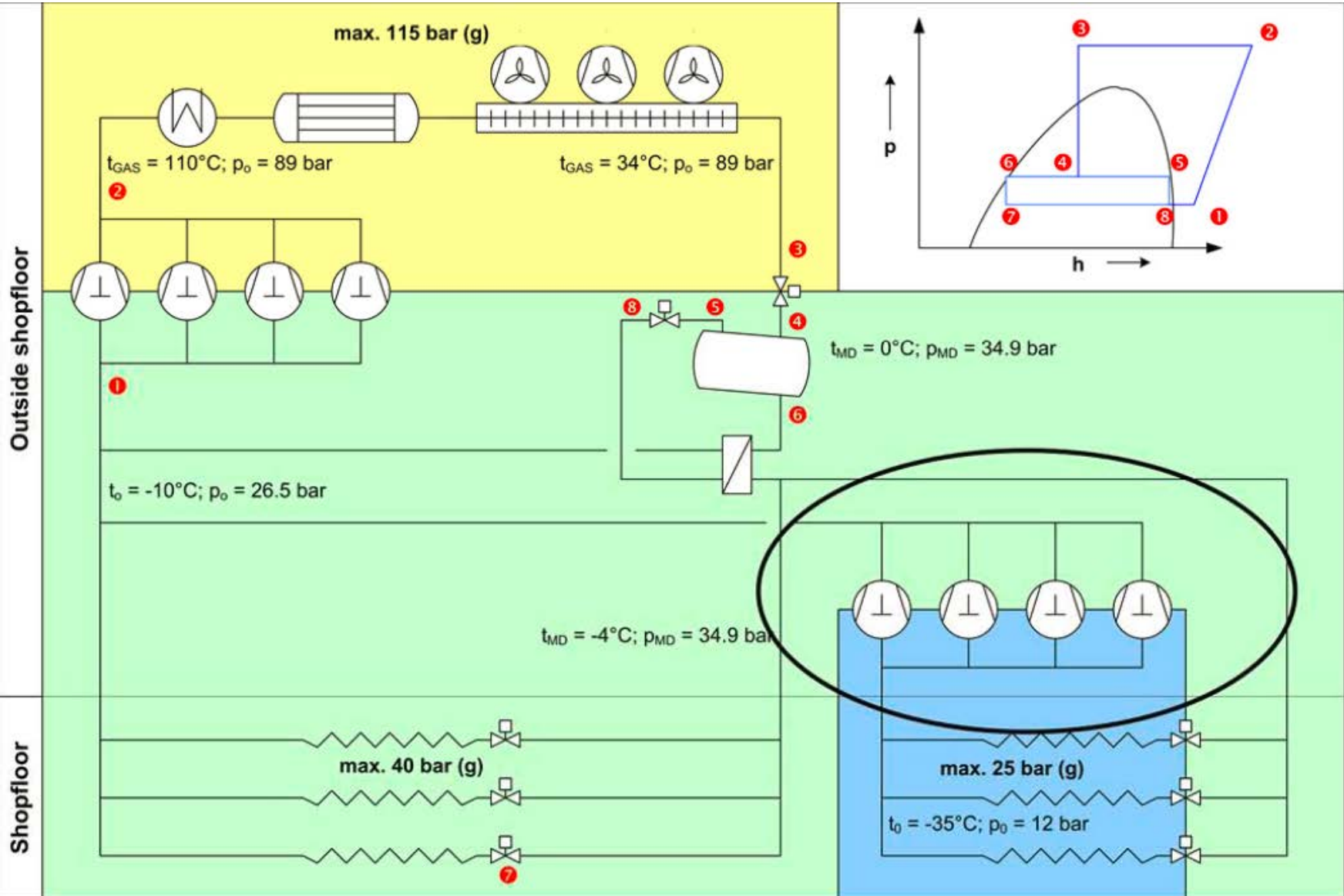


Source: adapted from Carrier, 2012

CO₂ system configurations



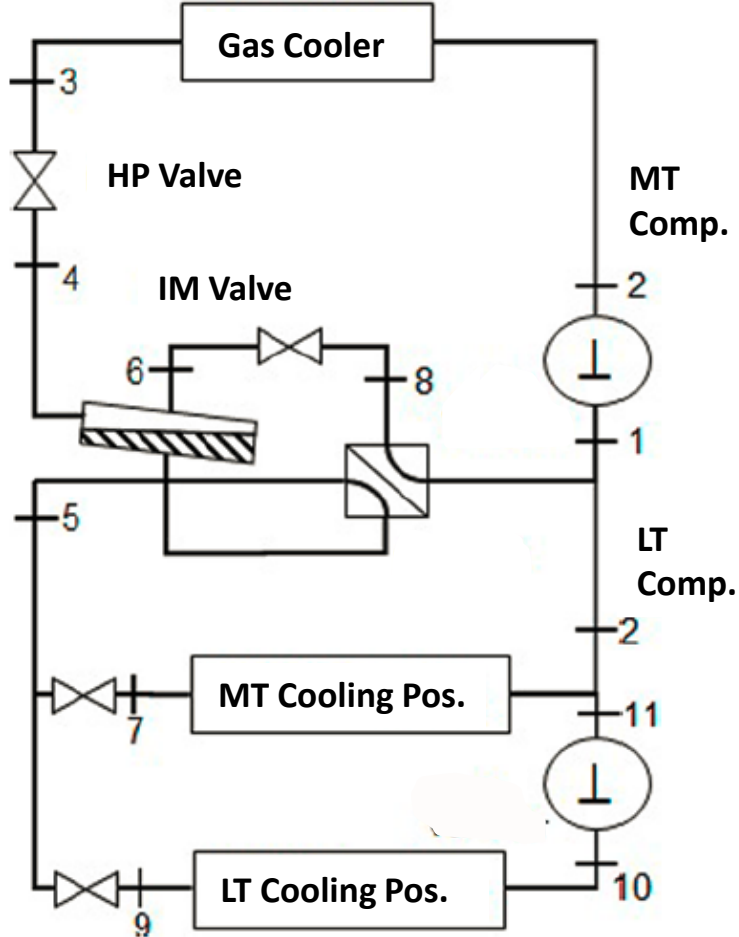
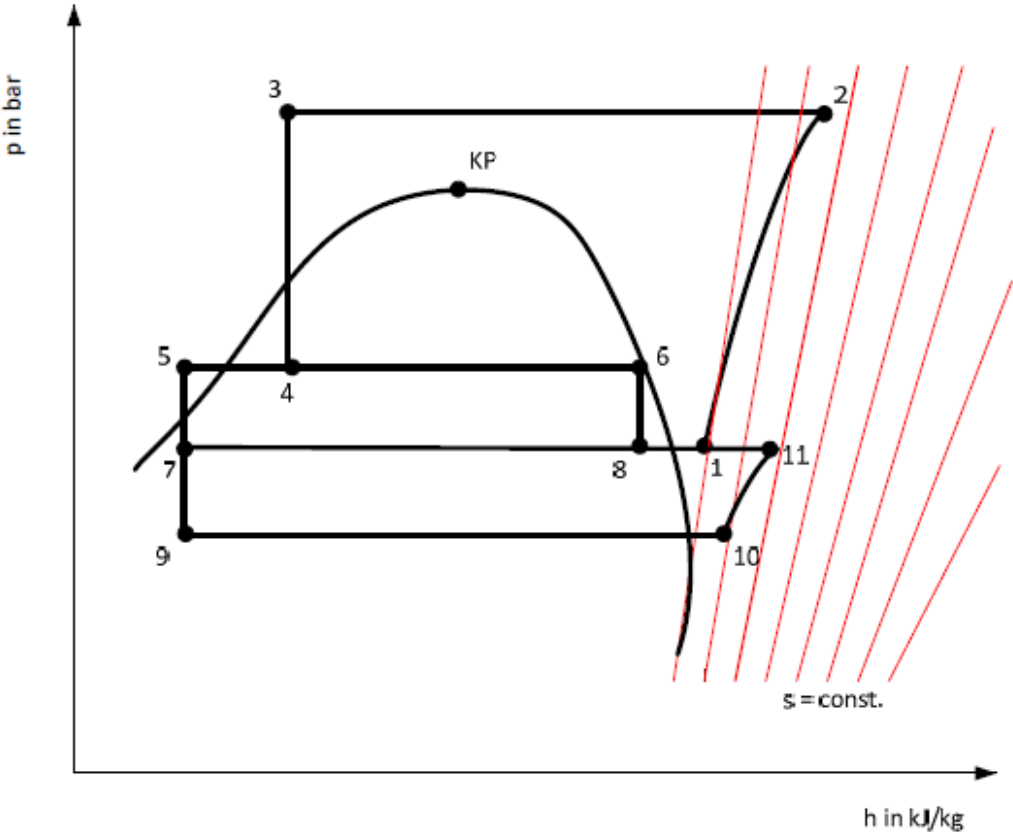
Supermarket booster system



CO² system configurations



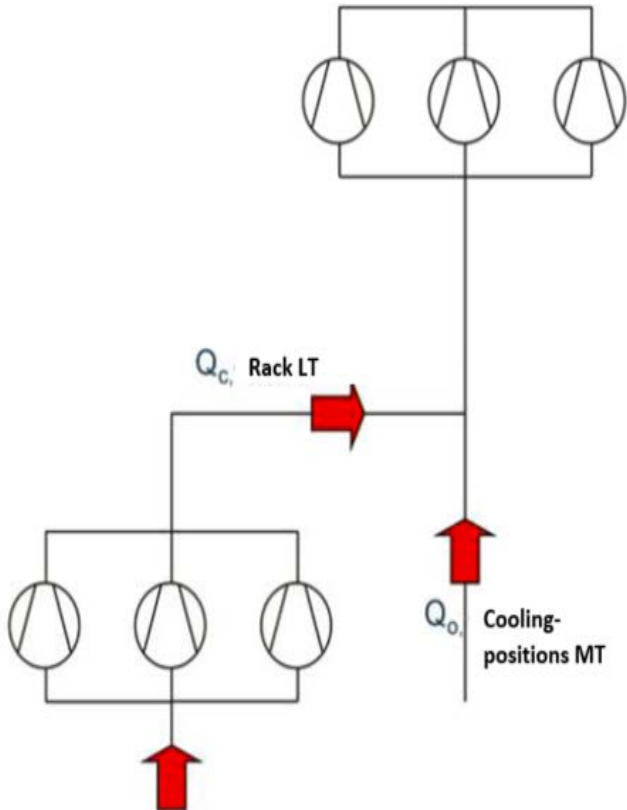
Supermarket booster system



CO² system configurations



Supermarket booster system

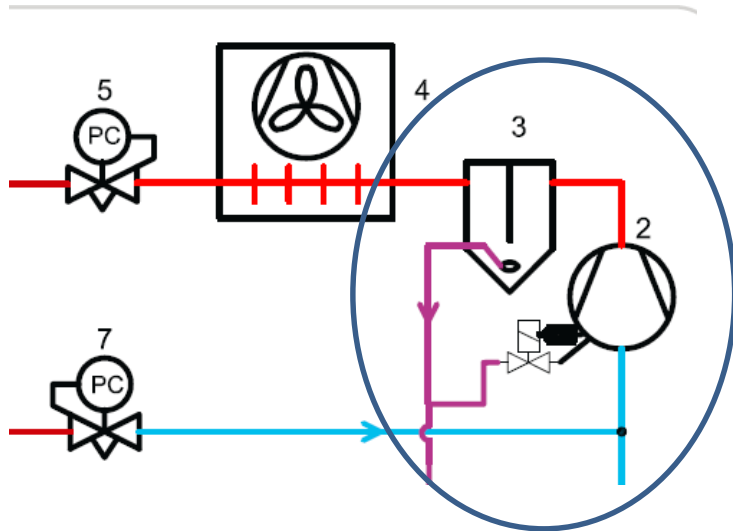


MT rack takes the hole LT condensing capacity:

$$Q_{o \text{ rack system (MT)}} = Q_{o \text{ cooling positions (MT)}} + Q_{c \text{ rack system (LT)}}$$

$$T_o \text{ (MT)} = t_c \text{ (LT)}$$

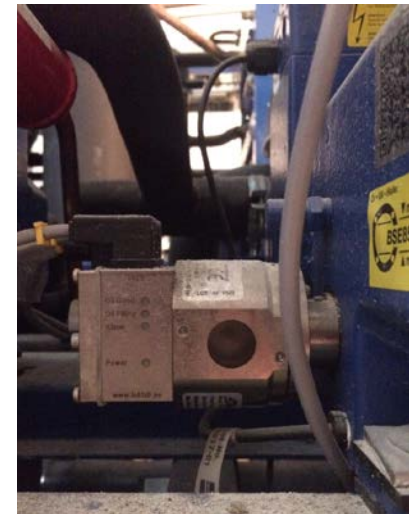
Supermarket booster system



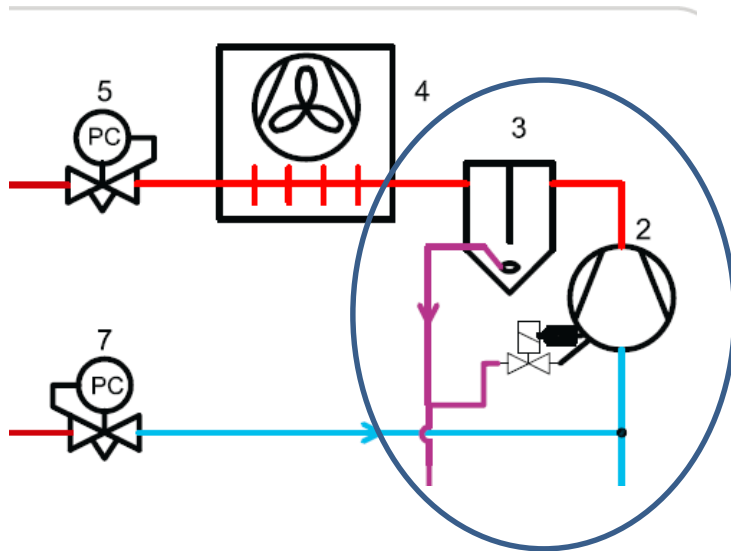
Oil charge = 4-6% of refrigerant charge

Oil management systems:

The oil separator and reservoir work on the same principles as a conventional oil management system (optical level switches), designed for the higher operating pressures (up to 125bar)



Supermarket booster system



Oil:

CO₂ Refrigeration oils are based on special synthetic esters, formulated for use in Carbon Dioxide (CO₂) compressors in all fields of industrial and commercial refrigeration.

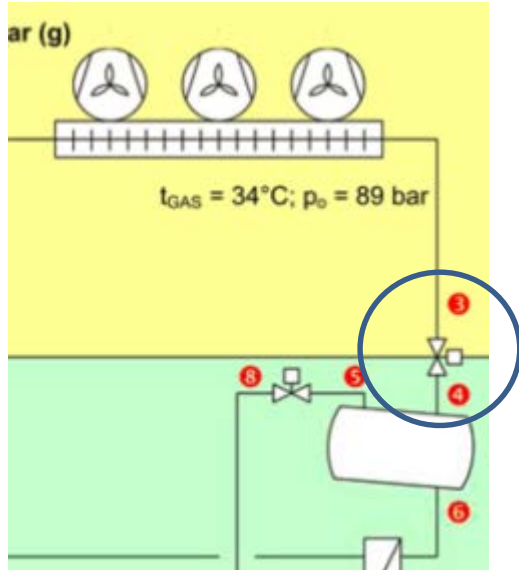
Some synthetic oil can be used in sub as well as trans-critical applications.

The oil needs to ensure optimum wear protection even under arduous CO₂ operating conditions.

Properties as high thermal and chemical stability, excellent deep-temperature flow-ability and good lubrication at high pressure levels.

Product name	ISO Viscosity mm ² /s	Pour point °C	Flash point °C	Water Content mg/kg	Colour index DIN ISO 2049	Sizes Available Litre	Refrigerants
RENISO C 55	55	-48	286	< 30	< 30	5	R744
RENISO C 85	85	-42	246	< 30	< 30	10	
RENISO C 170	170	-40	286	< 30	< 30	10	

Supermarket booster system

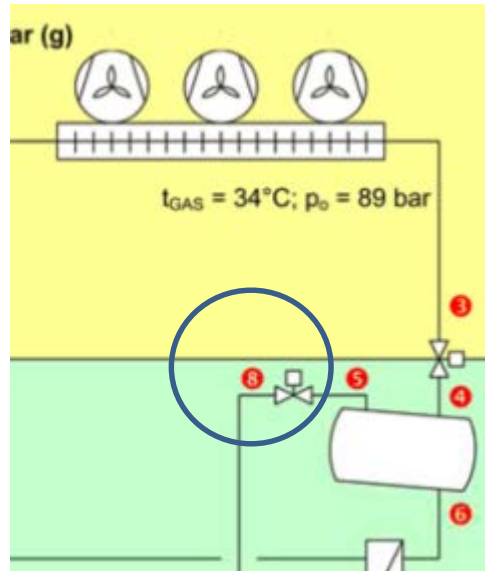


High pressure valve:

- Engine driven valve
- Operates like a expansion valve
- Analogue controlled (such as: 0-10V)



Supermarket booster system



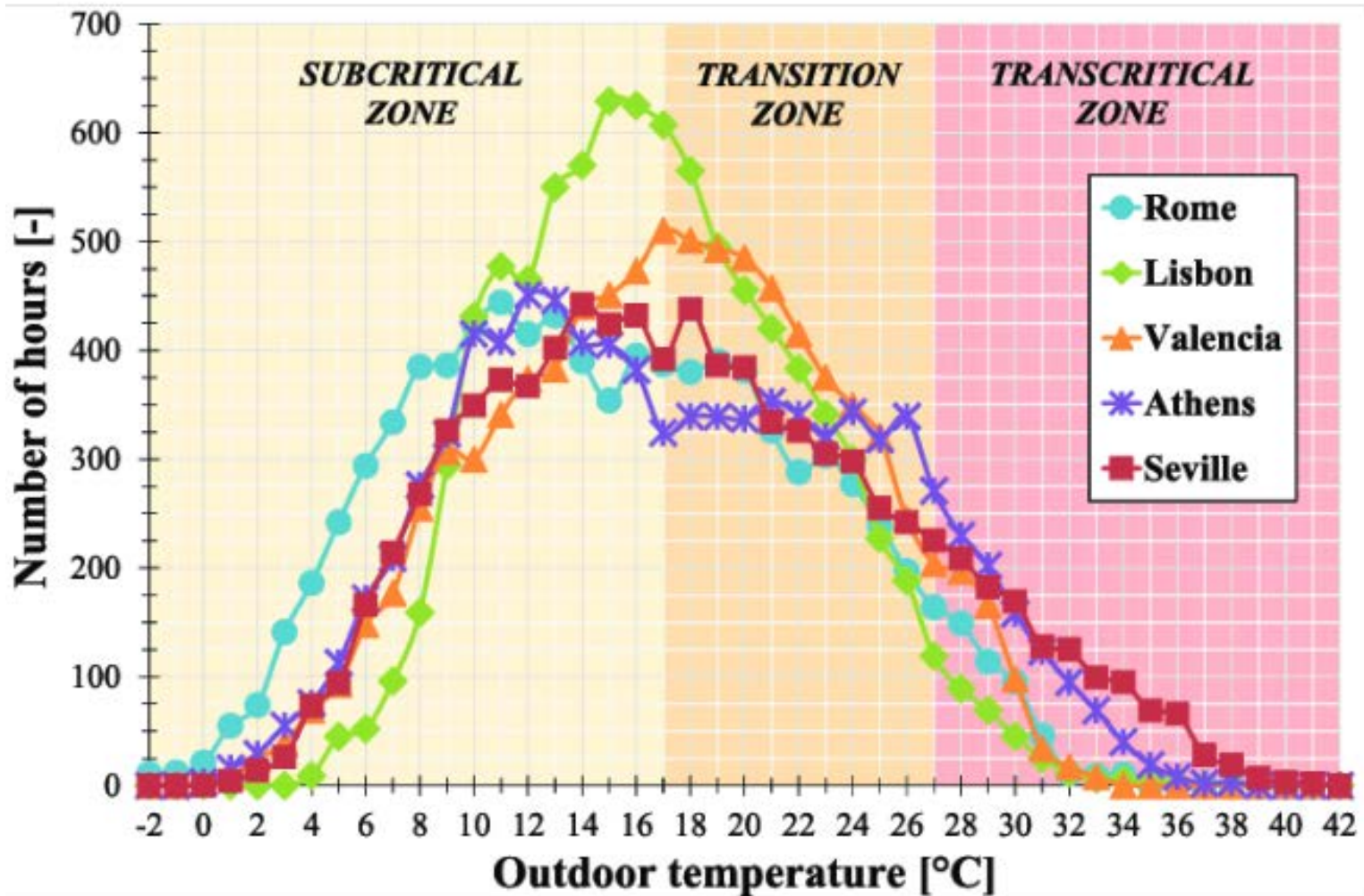
Intermediate pressure valve:

- Engine driven valve
- Operates like a expansion valve
- Analogue controlled (such as: 0-10V)



Ejector efficiency

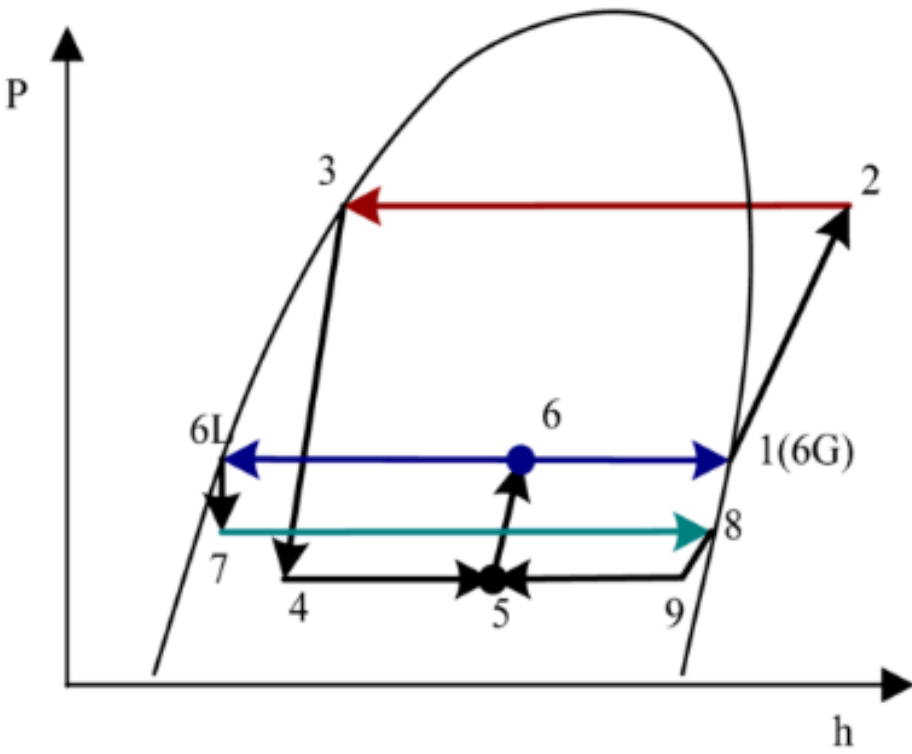
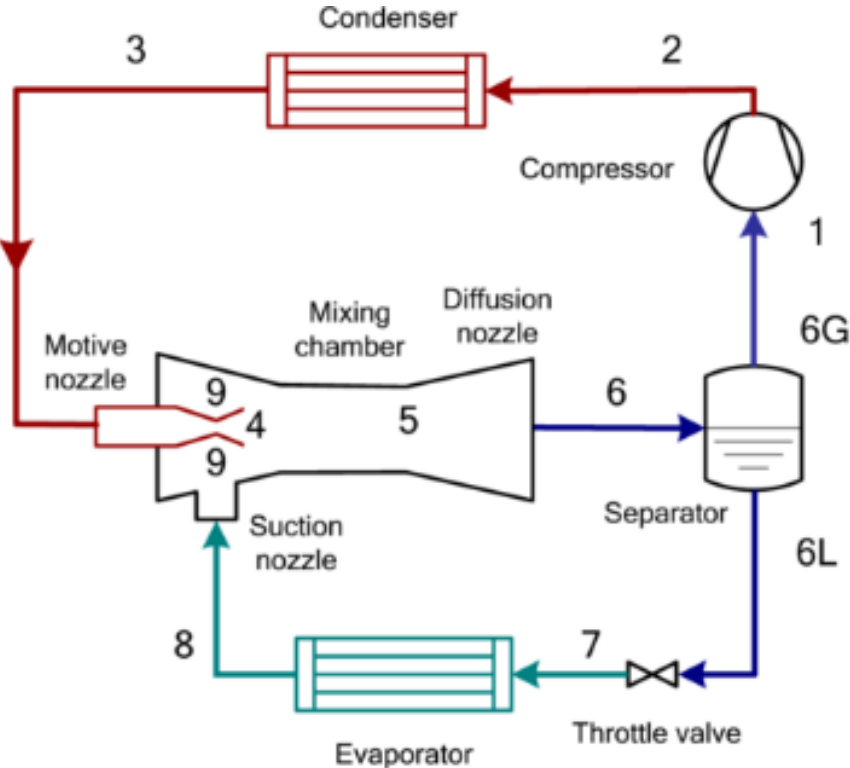
- Number of hours per year at different outdoor temperatures in the selected locations (Remund et al., 2014).



CO₂ system configurations



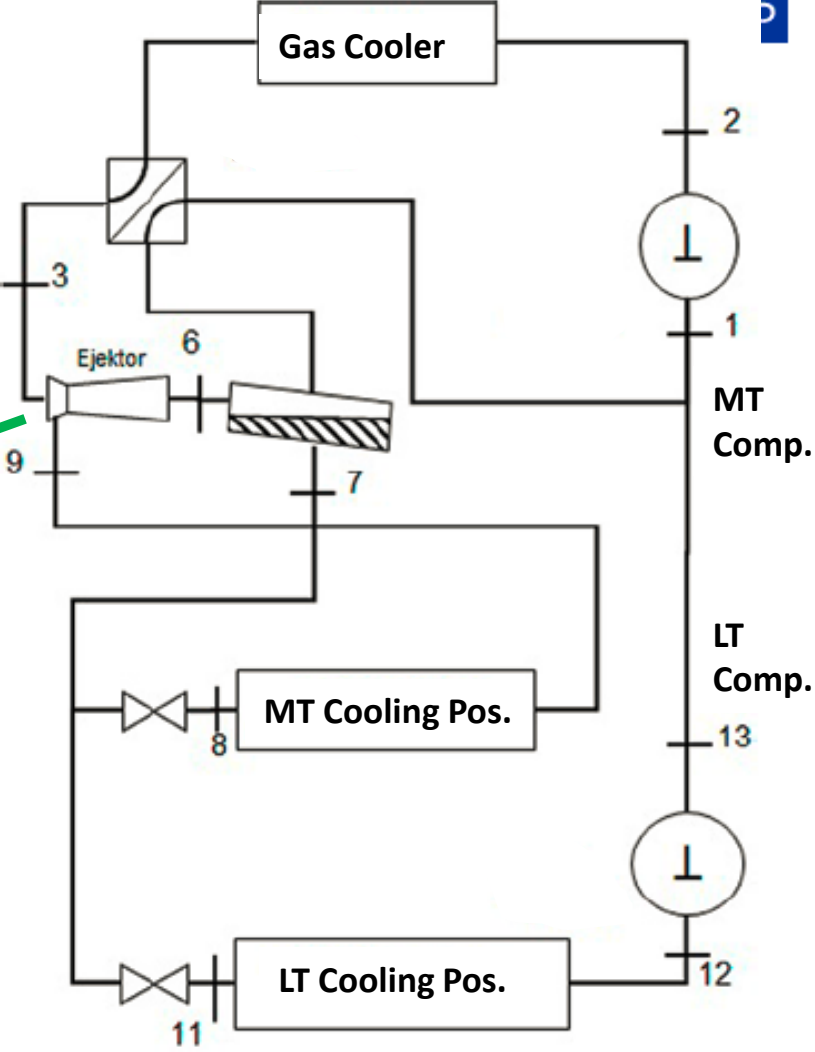
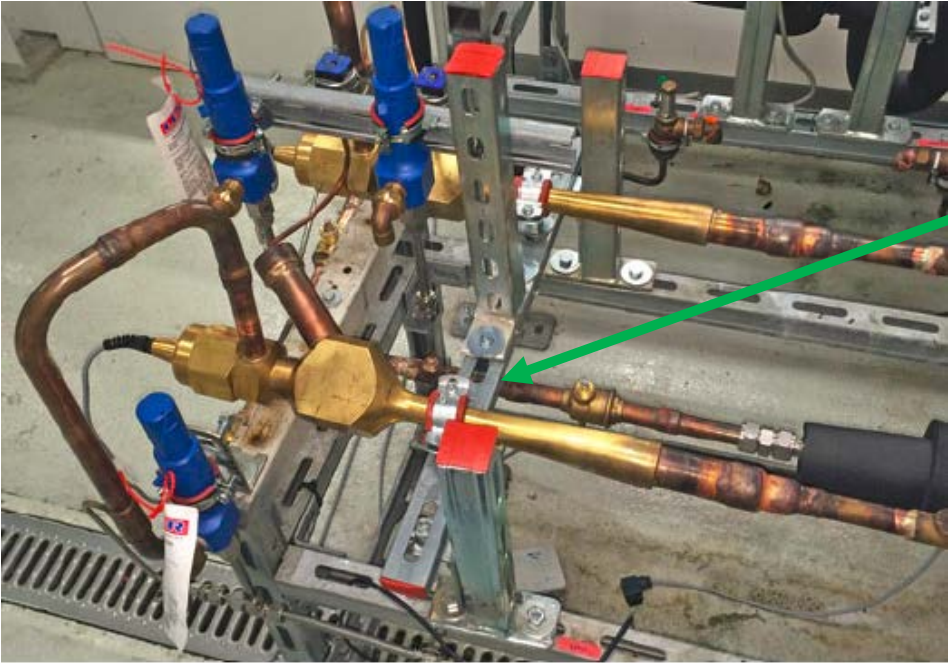
Ejector system



CO² system configurations



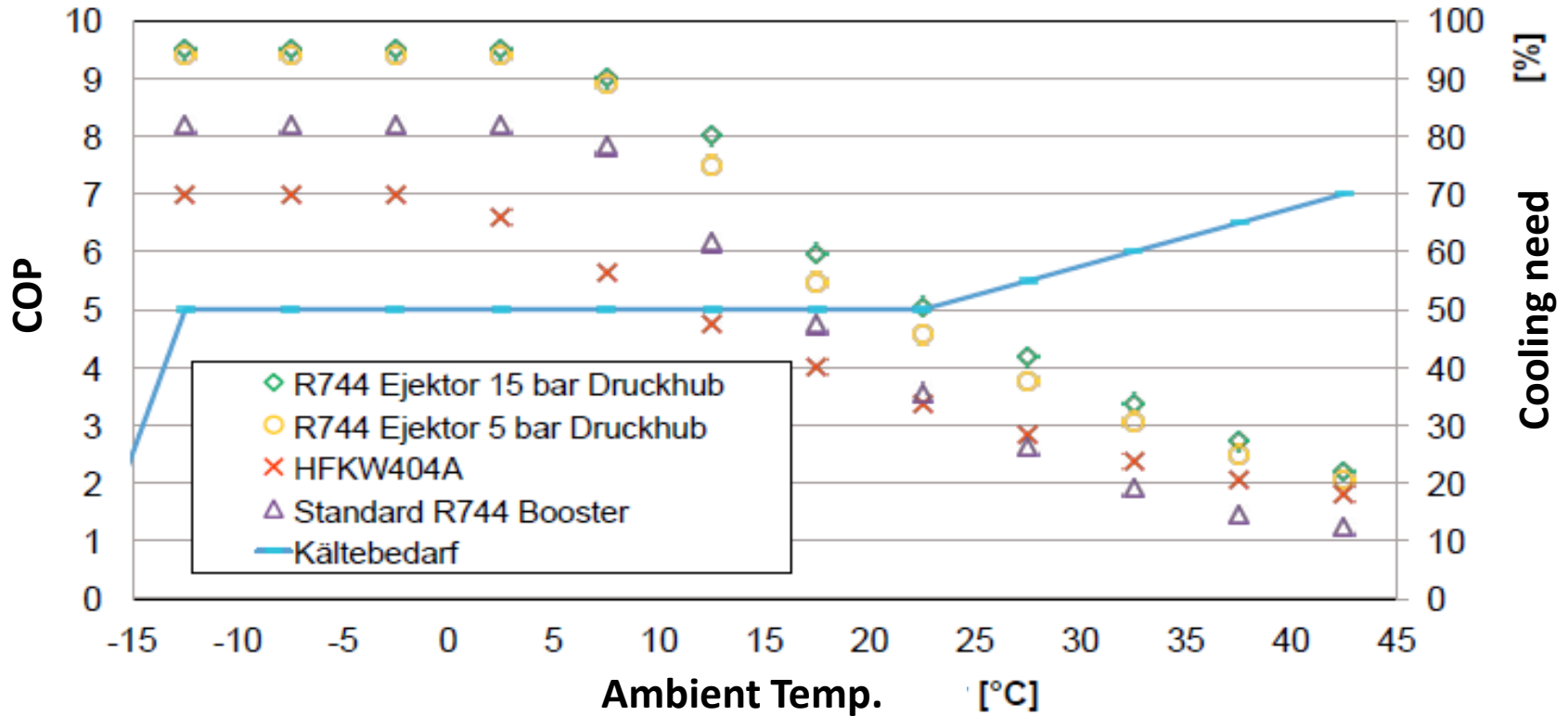
Ejector system



CO² system configurations



Ejector efficiency

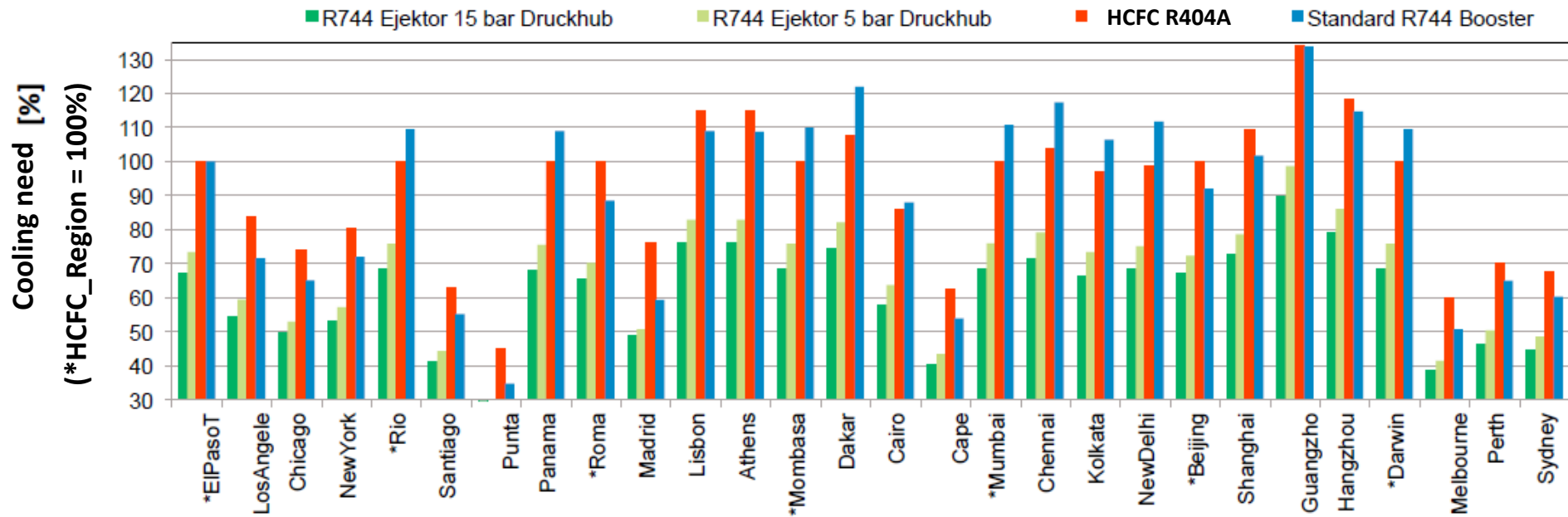


COP and cooling need of diff. system configurations relating to ambient temperature

CO² system configurations



Ejector efficiency

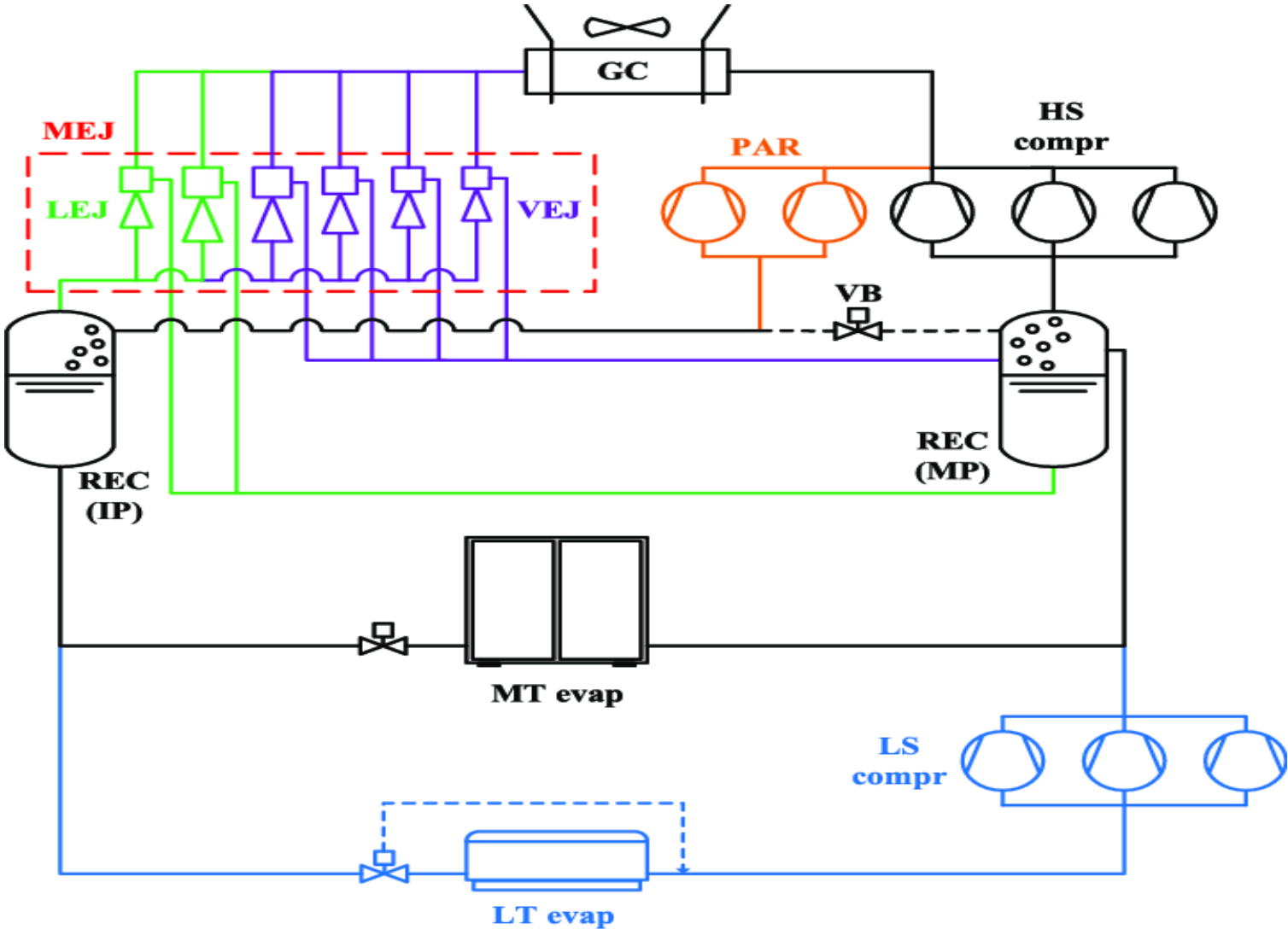


Annual energy requirement (relative) of commercial refrigeration systems in four different locations

CO² system configurations



Ejector system & parallel compression

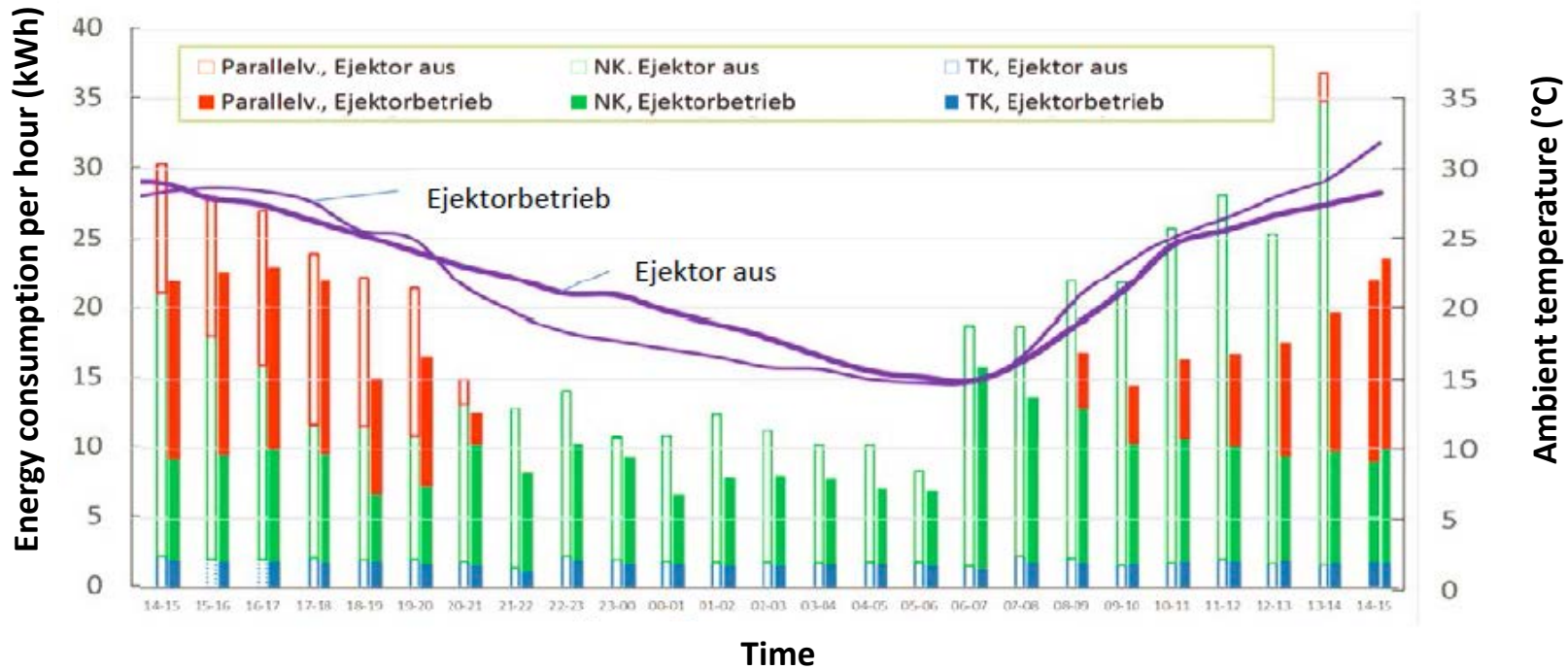


CO² system configurations

Ejector system & parallel compression



Ejector efficiency



Energy consumption of compressor groups per hour measured on 2 days in June (2015) -> 1 day with ejector / -> 1 day without ejector

Overview: Modules and Sections

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CO₂ application potential

Design, Construction, PPM

Safety issues

First Aid

Review of tools and equipment

Commissioning and maintenance

Mechanical integrity



Wear the adequate PPE for each activity!

Gloves and goggles for protection against freeze burns (as with all refrigerants)

Overall / long sleeves and legs

Safety shoes

Check: All necessary personal protective equipment is available and being used correctly.



Prohibitions



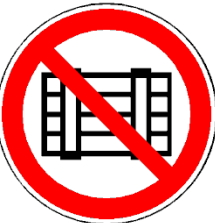
No smoking



No open flames / fire



Do not cross / walk through



No storage of materials / goods



Not for non-authorized persons

Warnings



Risk : presents of high concentration of CO_2



existence of pressurised cylinders with CO_2

Commandments



Wear safety goggles and canister respirator mask



Wear suitable working clothes and hard hat



Wear suitable working gloves



Wear ear protection

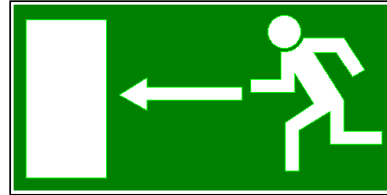


Disconnect from electrical mains when servicing

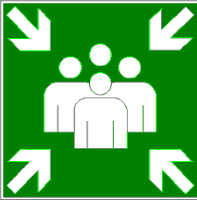
First Aid



First aid station



escape / rescue way



People assembly point



Place of water shower



Place of eye shower / equipment

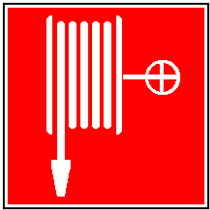


First aid / emergency telephone

Fire Protection



Place of fire extinguisher – CO₂ or powder



Place of fire hose – if applicable



Location sign



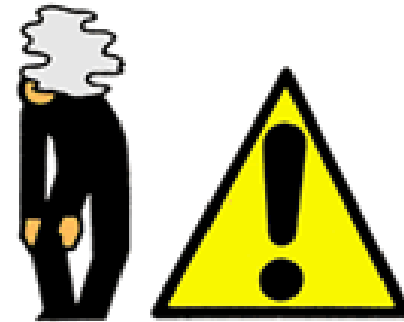
Fire fighting telephone

What are the Hazards of CO₂

The hazards of CO₂ should not be underestimated.

The main safety hazards are:

- Suffocation;
- High pressures;
- Rapid expansion of trapped liquid or gas;
- Low “boiling point”;
- Solidification.



Comparison of some of these hazards to those of HFC R404



	CO₂	R404A
Asphyxiation	PL* = 0.1 kg/m ³ Odourless Heavier than air	PL* = 0.48 kg/m ³ Odourless Heavier than air
Pressure at 15°C	51 bar g	8.5 bar g
Boiling point at atmospheric pressure	-78.5°C	-47°C
Solidification	Possible	Not possible

PL* is the practical limit specified in EN378 (Refrigerating systems and heat pumps - safety and environmental requirements). It is the highest concentration level in an occupied space which will not result in “escape impairing effects”.

Suffocation



The practical limit of CO₂ is significantly lower than HFCs/HCFC to reflect the different physical effects CO₂ has when inhaled:



CO ₂ concentration in air		
Vol %	ppm	Info
0,0368	386	CO ₂ concentration in the atmosphere
0,5	5.000	maximum allowable concentration (MAC) at workspace
1,5	15.000	Pre - alarm (Gaswarning system)
2	20.000	Main- alarm (Gaswarning system)
3-> 5	30.000-> 50.000	Headache; disruption of respiration; discomfort
5,4	54.000	Practical limit (PL) En 378-1
8-> 10	80.000-> 100.000	Cramps; black out; shortness of breath; high frequent heart beat lead to unconsciousness, can lead to death in 30-60 min.
20	200.000	Lead to death within 5-10min.

CO₂ is Invisible, Silent, there is No Smell and than it Kills



Main symptoms of Carbon dioxide toxicity

Volume % in air	
■	- 1%
■	- 3%
■	- 5%
■	- 8%

Visual

- Dimmed sight

Auditory

- Reduced hearing

Central

- Drowsiness
- Mild narcosis
- Dizziness
- Confusion
- Headache
- Unconsciousness

Skin

- Sweating

Heart

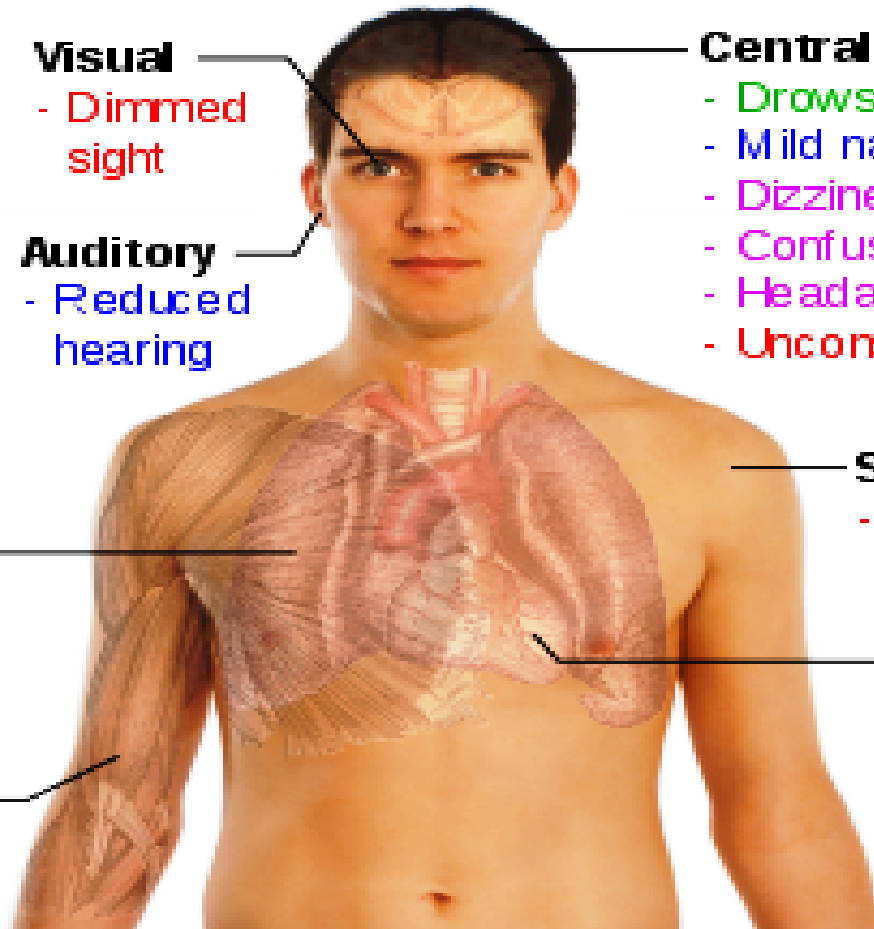
- Increased heart rate and blood pressure

Respiratory

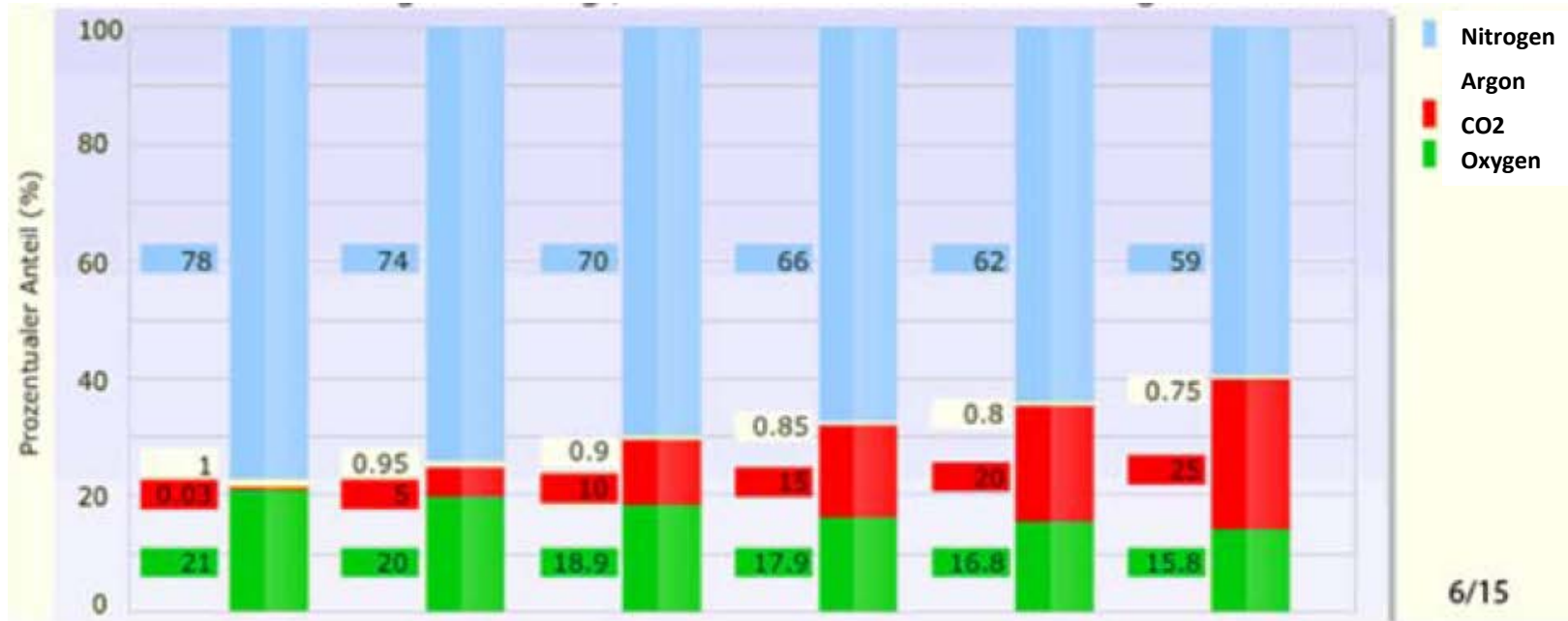
- Shortness of breath

Muscular

- Tremor



Suffocation



The steady increase of CO₂ increasingly displaces argon and nitrogen, while the oxygen content remains almost constant.

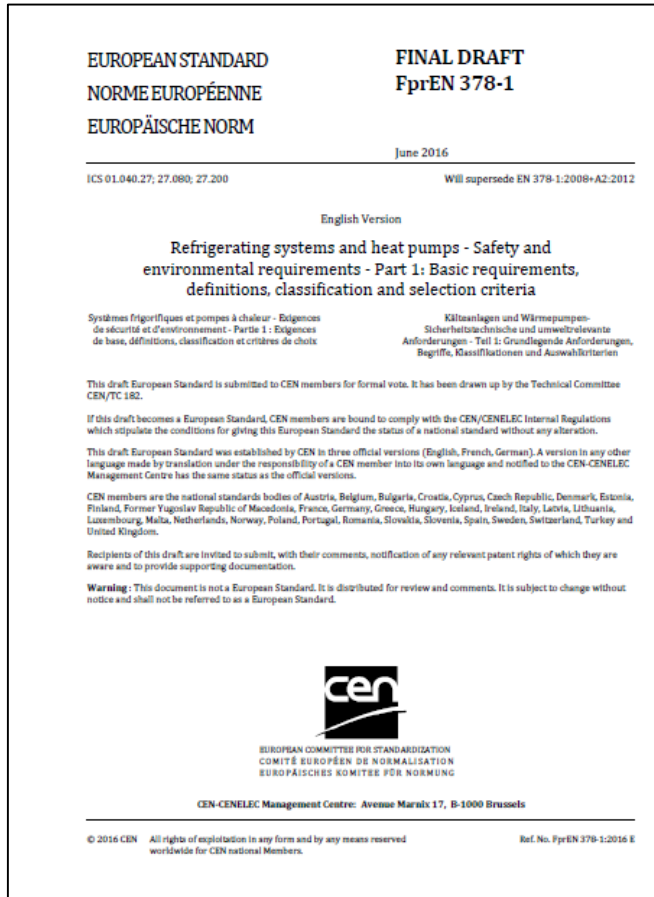
!! Never use oxygen detector to indicate CO₂

Avoid Suffocation with a number of measures



- Areas around CO₂ systems should be very well ventilated
- If you feel any of the warning signs of suffocation (dizziness, numbness, tingling in fingers) you should move into the fresh air as quickly as possible.
- A personal gas detector should be used in working areas.
- Gas detection should be located in any area where a leak of CO₂ would exceed the practical limit (0.1 kg/m³).
- This should be fitted at low level, 30cm above the floor. These sense CO₂, not oxygen deficiency, and typically alarm at 1%.
- Do not enter a room if the gas detector is alarming.
- CO₂ is vented from systems rather than recovered – ensure it is vented into a very well ventilated area or to outside.
- Wear ear defenders when venting - this is a very noisy process.





CO2 is suffocating like HCFC and HFC refrigerants

CO2 = Safety classification A1

EN 378-3 (9.1)

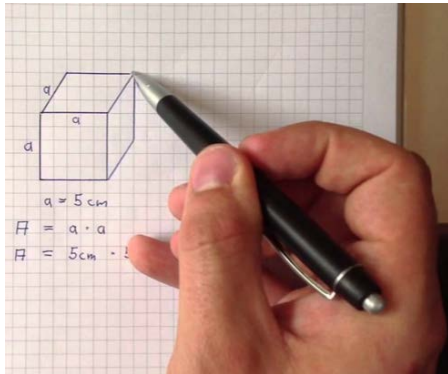
When the concentration of the refrigerant can exceed the practical limit in accordance with EN 378-1:2016, Annex C, detectors shall at least actuate an alarm and in the case of the machinery room the emergency mechanical ventilation.

PL of R404A = 0,48 kg/m³

PL of R134a = 0,25 kg/m³

PL of R744 = 0,10 kg/m³

Room calculation for gas detection



Room = L 10m x W 20m x H 3m

Volume = 600 m^3

Charge size of the system = 120kg CO₂

$0,1 \times 600 = 60 \text{ kg}$

If charge is higher than 60kg a gas detection needs to be install!

Alarm : equal or less than 50% of practical limit (PL)

Gas detection



Position gas sensor close to rack or evaporators where leaks could occur and refrigerant could accumulate (EN378-3:2016 article 9.2)

Gas sensor to be fitted 20 cm to 30 cm above floor level; (CO₂ is approx. 1.8 times heavier than air; EN378-3:2016 art. 9.3)

Gas alarm has to be robust, contain self-test function and set points must not be adjustable by unauthorized people (EN378-3:2016 article 9.4)

Before starting service:

- Check working conditions
 - >Do not work in confined spaces
 - >If not preventable: 2. technician as safeguard outside
- As for working permission and inform customer when service job is done.
- Check Gas alarm system before entering controlled areas
- Always ensure proper ventilation while working on CO2 refrigeration cycle.




In case of gas alarm:

- Leave the affected area immediately
- Inform persons around
- Ensure proper ventilation
- Enter the affected area only if gas alarm is off and gas alarm system shows normal operation
- In case persons need to be rescued, only entering the room by using Self Contained Breathing Apparatus


First Aid:

Always follow the first aid rules according safety data sheet!

 IFA
Institute for Occupational Safety and Health
of the German Social Accident Insurance

GESTIS Substance Database

Carbon dioxide



IDENTIFICATION

Carbon dioxide
Air fixe
Carbonic acid
Carbonic acid gas
Carbonic anhydride

ZVG No: 1120
CAS No: 124-38-9 carbon dioxide
EC No: 204-696-9

Related
CAS No: 463-79-6 carbonic acid

CHARACTERISATION

SUBSTANCE GROUP CODE

125100 Carbon oxides
139100 Inorganic gases
120510 Acids, inorganic

STATE OF AGGREGATION

The substance is gaseous.

PROPERTIES

colourless
odourless

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Avoid Problems Associated with High Pressure



- The equipment used to pressure test, charge, vent and measure pressures must be suitable for the pressures of CO₂.
- A standard gauge set must not be used.
- The equipment must not allow refrigerant to be trapped between closed valves.
- Take great care when accessing a system, even if it is off.
- Ensure cylinders are secure (ideally in a suitable trolley) and open them slowly - the pressure can de-stabilise the cylinders if they are opened quickly.



Manifold set (4-Way recommended)



Commissioning:

CO2 manifold 120bar is needed

-> strength test ($P_{sx1,1}$)

Service:

Some electronic manifolds (incl. R744) are available for pressure up to 60bar.

(Intermediate pressure)



Portable gas sensor: how to use it



While technicians are working on the system, it can be advisable to use a portable gas detector.

- Detector can be clipped to clothing or placed on the floor within the working area.
- It should be switched on for the duration of the work and set to alarm at 50% of the PL, to alert staff members
- For safety reasons, the gas detector should not have a “Zero Background” function.
- Do not use Oxygen detectors as PPE
- Technicians are capable of immediately acting on the relevant emergency procedures.



Standard vacuum pump
(electrical connections
and sealing)

Do not use “home-made”
vacuum pumps



There is no recovery process, due to high pressure!

Refrigerant venting to the environment!

Take care to avoid oil contamination to the environment!



Oxygen free dry nitrogen (OFDN)



Must have for the applications:

- Pressure / strength test
- Equipment flushing
- Inert gas brazing



Cylinder Content (Litre)	Filling Pressure (bar)	Gas Content (m ³)	Cylinder gross weight (ca. kg)
5	200	1	9,8
10	200	1.911	15,7
20	200	3.822	37,0
50	200	9,556	77,7

Purity 4.0 OFDN gas is of 99,99 % and a water content of maximum of 30 ppm.

For refrigerant charge

Measuring range ≤ 100 kg

Accuracy of about $\pm 0.5\%$

! You are only able to
empty the cylinder until
intermediate pressure
(max. suction pressure)



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High coefficient of expansion



In addition to the hazard caused by the very high standstill and operating pressures, trapped CO₂ liquid or gas expands rapidly.

This causes an extremely high pressure increase.

For example, if liquid is trapped in an evaporator at -30 °C, its pressure will rise to over 100 bar/g if its temperature rises to -20 °C.

**As a rule of thumb
for every 1°C rise in
temperature, the pressure
of trapped CO₂ liquid rises
by 10 bar.**



All pressure level of CO2 systems need to be secured against excessive pressure.

Due to the extremely high pressure increase, CO2 systems do have safety relive valves at all pressure levels.

Example of pressure levels:

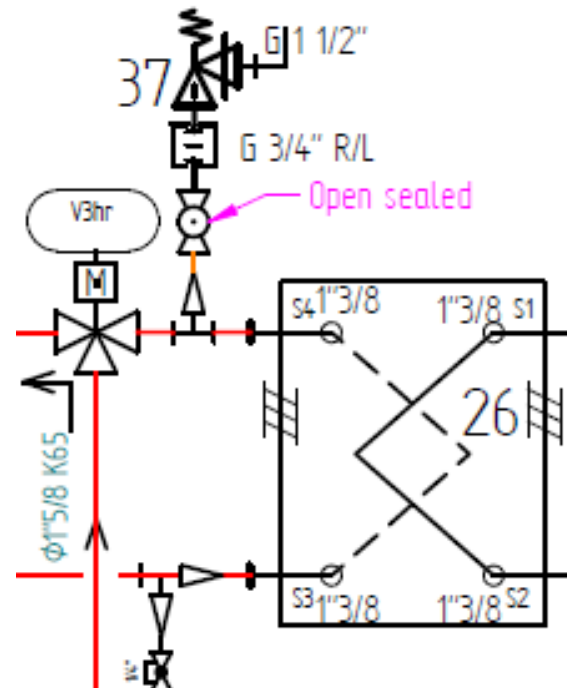
High pressure	= 120bar
Intermediate pressure	= 45bar
MT cooling positions	= 45bar
High pressure LT	= 45bar
LT cooling positions	= 25bar

We differentiate into 2 types of safety valves:

- Main safety valves
- Secondary safety valves

Main safety valves

- Necessary at each pressure level
- Vent line always connected to the outside
- Always activated not lockable (without tools)

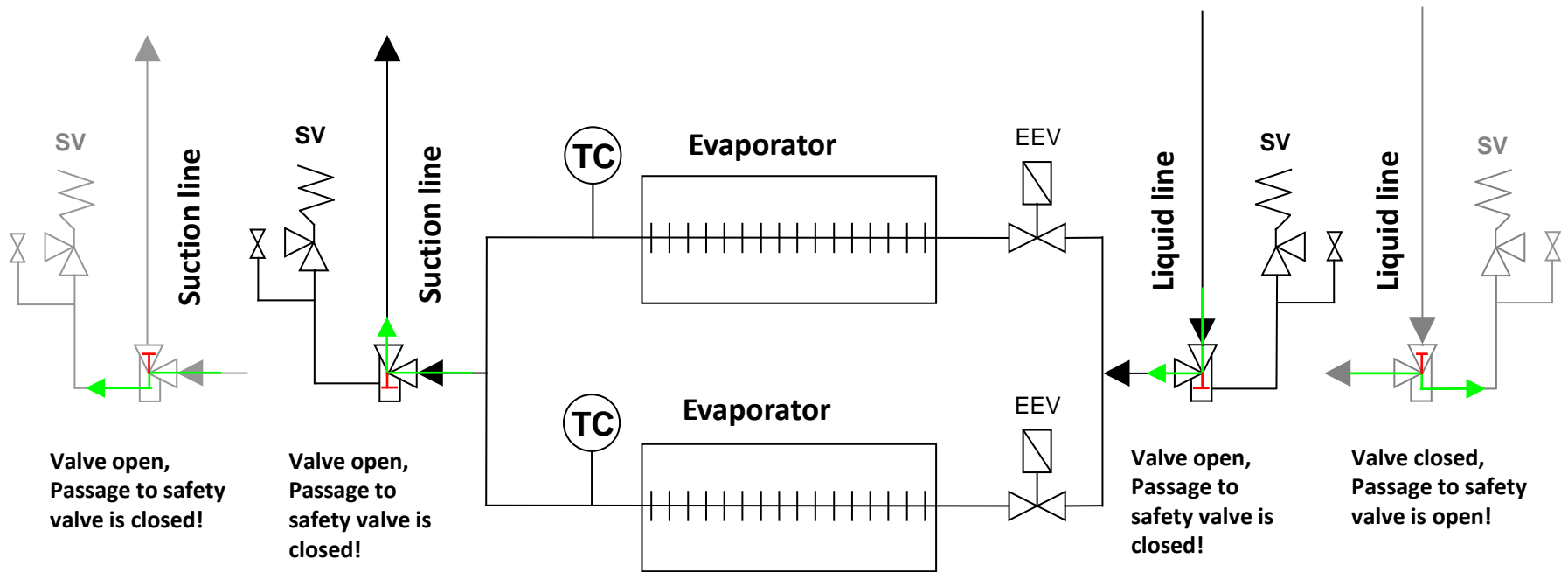


Secondary safety valves

- Necessary for service safety
- Vent line to surrounding
- Only activated in case of service



Secondary safety valves



Dry Ice Formation



Solid CO₂ (dry ice) is formed at certain conditions when the pressure or temperature of CO₂ is dropped.

This can occur when charging CO₂ into an evacuated system or when CO₂ is vented.

The pressure drops, but when the dry ice turns to gas (sublimes) the pressure rises rapidly.

Dry ice has a very low surface temperature.



Avoid problems associated with rapid expansion of trapped CO₂ in the system:



- Do not allow liquid to be trapped in pipes or components between closed valves.
- Do not weld or braze pipe work or components which contain CO₂.
- Ensure valves are open, for example by using a magnet to open solenoid valves.

Preventing dry ice formation

- To prevent dry ice formation when charging, charge gas until the pressure in the system is at least 4.2 bar g (i.e. above the triple point), then charge liquid to complete the charge.
- To prevent dry ice formation when venting the refrigerant should be vented in the liquid state if possible.
- Be aware that if dry ice does form the pressure drops to atmospheric pressure, and will rise again when the dry ice sublimates.



System charging

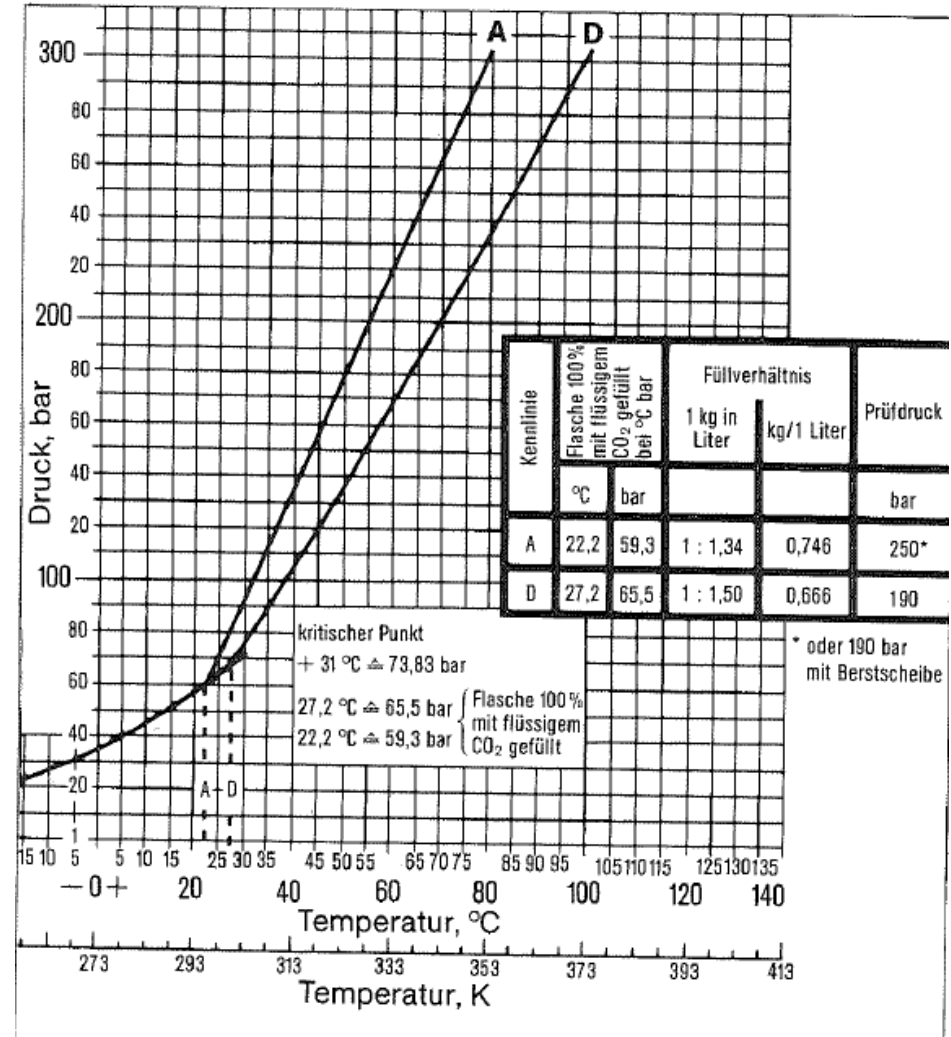


R744 cylinders are high pressure cylinders!

-> at hot day`s cylinders can easily reach 100bar

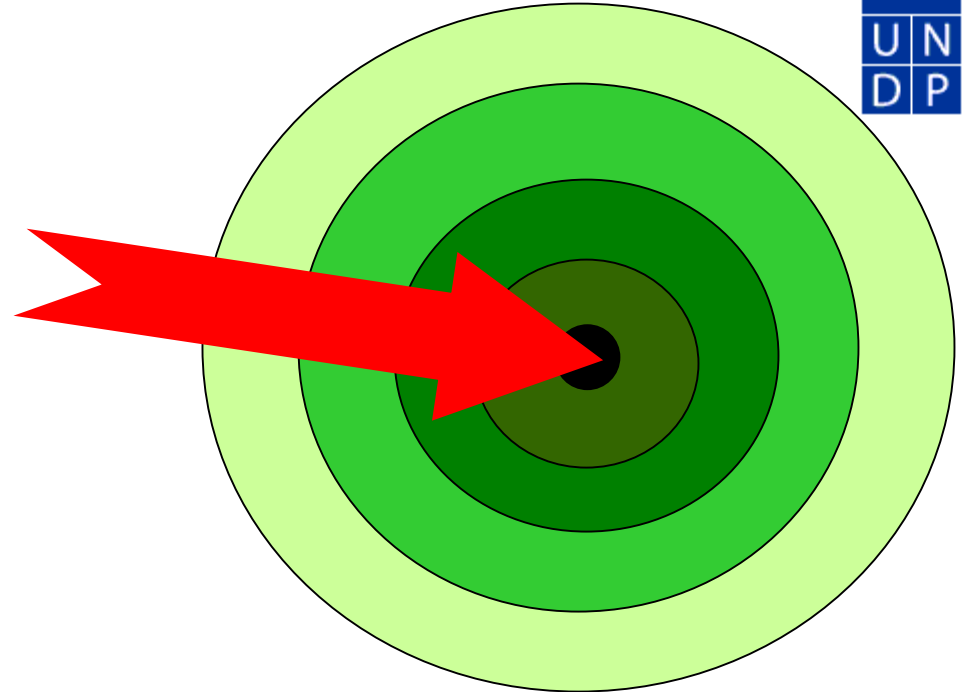
! Only use approved tools!

-> at cold day`s cylinder pressure can be lower than system pressure





Targets



no Accident

no Environment Pollution

no Health Hazard



Thank You for Your attention!